



VRF Component

Compiled referring to training material in JCH

JCH-WX VRF-Com
Hitachi-Johnson Controls Air Conditioning

The date of publication: Feb 20th 2019

Version: 1.00



VRF Component

LEARNING OBJECTIVES

- Component Identification
- Refrigerant Cycle
- Compressor and Control
- Fan control and Fan RPM
- SVA,SVB,SVF,SVX
- Sub cooling heat exchanger

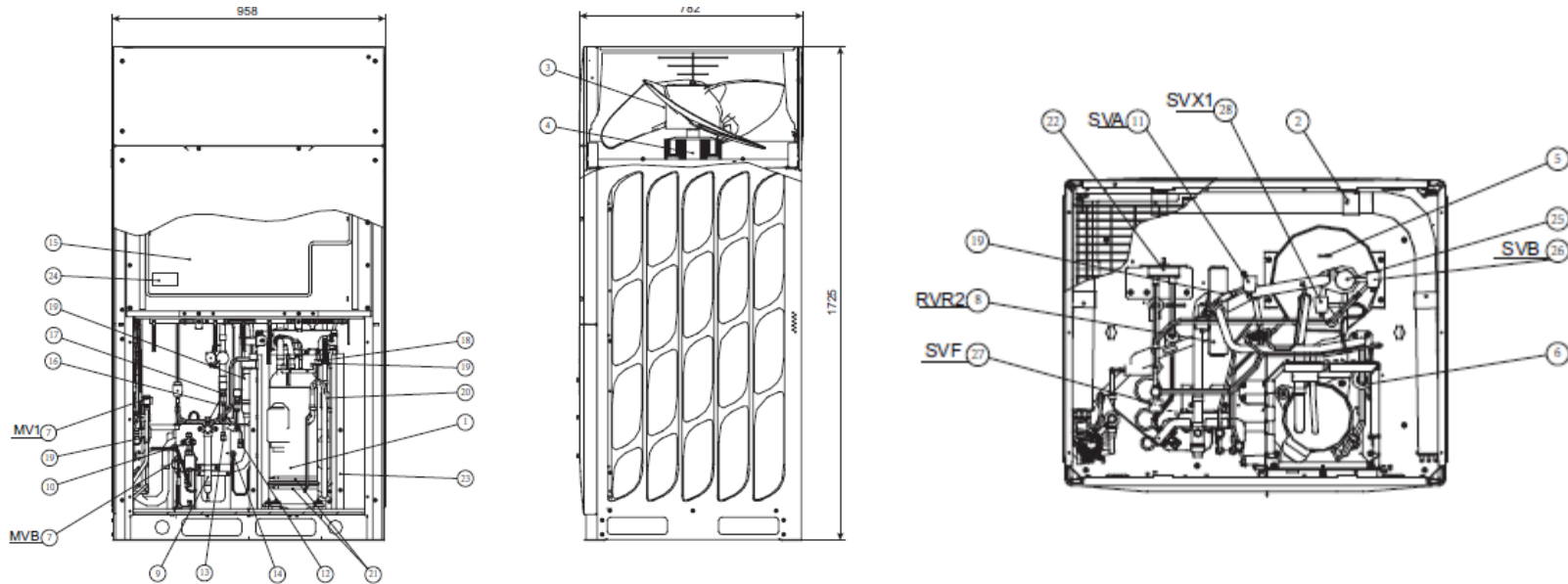


VRF Component

Component Identification

■ Model RAS-8.0-12HNBCM

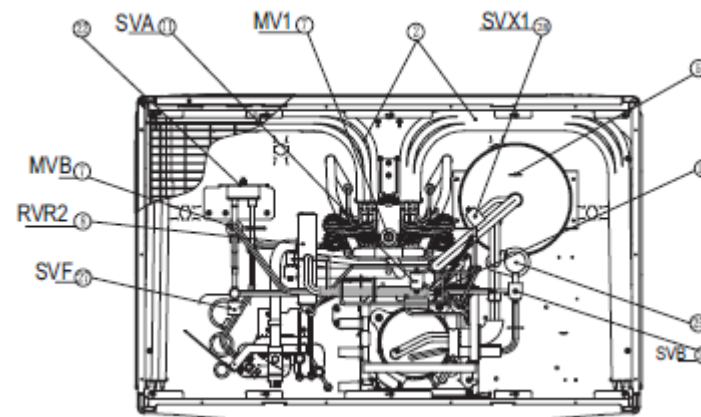
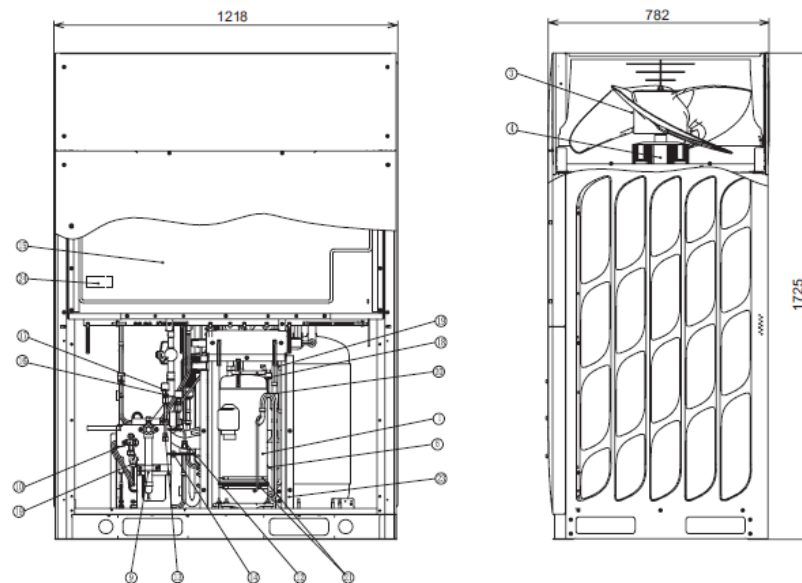
Unit: mm



Item	Part Name	Item	Part Name
1	Compressor (Inverter)	15	Electrical Control Box
2	Heat Exchanger	16	Low Pressure Sensor
3	Propeller Fan	17	High Pressure Sensor
4	Fan Motor	18	High Pressure Switch for Protection
5	Accumulator (Pressure Vessel)	19	Strainer
6	Oil Separator (Not Pressure Vessel)	20	Check Valve
7	Micro-computer Controlled Expansion Valve (2 pcs)	21	Crankcase Heater (2pcs)
8	4-Way Valve	22	Plate Heat Exchanger
9	Stop Valve (Gas)	23	Compressor Cover
10	Stop Valve (Liquid)	24	Terminal Block Box
11	Hot Gas Bypass Solenoid Valve (SVA)	25	Injection Muffler
12	Check Joint (Low)	26	Injection Solenoid Valve (SVB)
13	Check Joint (High)	27	Injection Solenoid Valve (SVF)
14	Check Joint (For Oil)	28	Solenoid Valve (SVX1)

■ Model RAS-14-16HNBCMQR

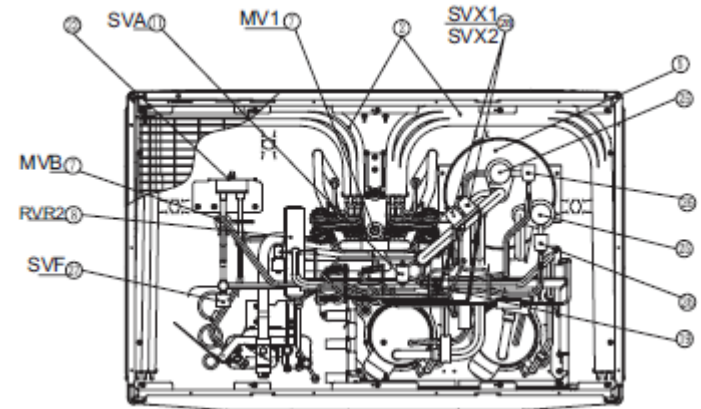
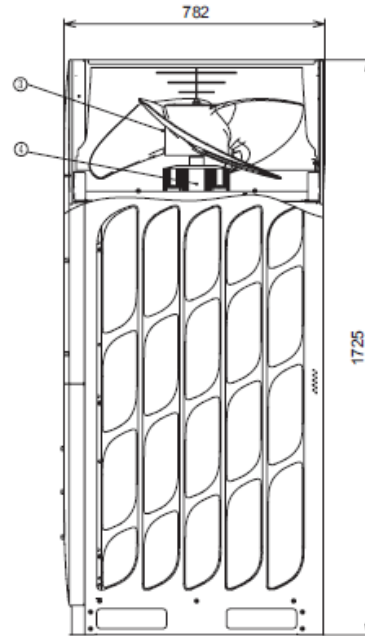
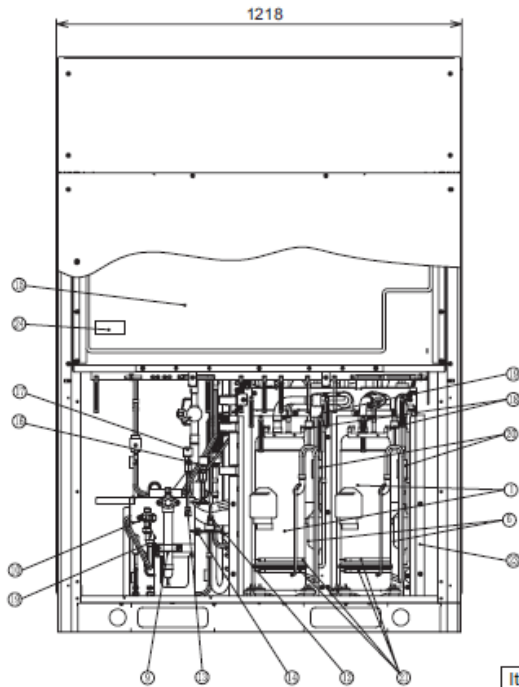
Unit: mm



Item	Part Name	Item	Part Name
1	Compressor (Inverter)	15	Electrical Control Box
2	Heat Exchanger	16	Low Pressure Sensor
3	Propeller Fan	17	High Pressure Sensor
4	Fan Motor	18	High Pressure Switch for Protection
5	Accumulator (Pressure Vessel)	19	Strainer
6	Oil Separator (Not Pressure Vessel)	20	Check Valve
7	Micro-computer Controlled Expansion Valve (2 pcs)	21	Crankcase Heater (2pcs)
8	4-Way Valve	22	Plate Heat Exchanger
9	Stop Valve (Gas)	23	Compressor Cover
10	Stop Valve (Liquid)	24	Terminal Block Box
11	Hot Gas Bypass Solenoid Valve (SVA)	25	Injection Muffler
12	Check Joint (Low)	26	Injection Solenoid Valve (SVB)
13	Check Joint (High)	27	Injection Solenoid Valve (SVF)
14	Check Joint (For Oil)	28	Solenoid Valve (SVX1)

■ Model RAS-18HNBCMQR

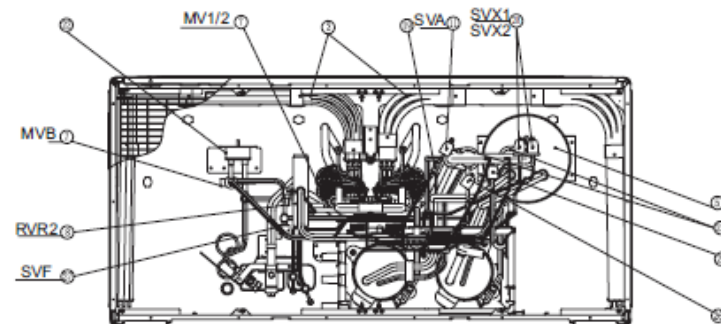
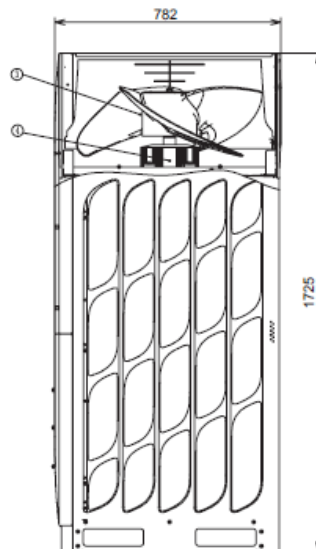
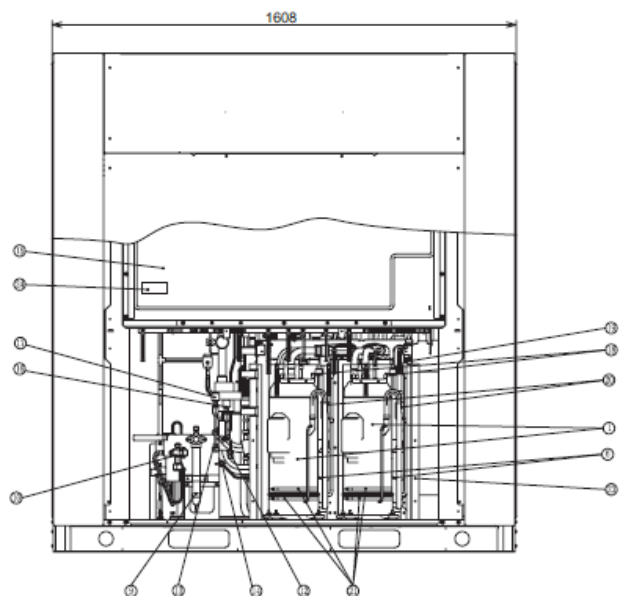
Unit: mm



Item	Part Name	Item	Part Name
1	Compressor (Inverter) (2pcs)	15	Electrical Control Box
2	Heat Exchanger	16	Low Pressure Sensor
3	Propeller Fan	17	High Pressure Sensor
4	Fan Motor	18	High Pressure Switch for Protection
5	Accumulator (Pressure Vessel)	19	Strainer
6	Oil Separator (Not Pressure Vessel) (2 pcs)	20	Check Valve
7	Micro-computer Controlled Expansion Valve (2 pcs)	21	Crankcase Heater (4 pcs)
8	4-Way Valve	22	Plate Heat Exchanger
9	Stop Valve (Gas)	23	Compressor Cover
10	Stop Valve (Liquid)	24	Terminal Block Box
11	Hot Gas Bypass Solenoid Valve (SVA)	25	Injection Muffler(2pcs)
12	Check Joint (Low)	26	Injection Solenoid Valve (SVB/SVC)
13	Check Joint (High)	27	Injection Solenoid Valve (SVF)
14	Check Joint (For Oil)	28	Solenoid Valve (SVX1/SVX2)

■ Model RAS-20-24HNBCMQR mm

Unit:



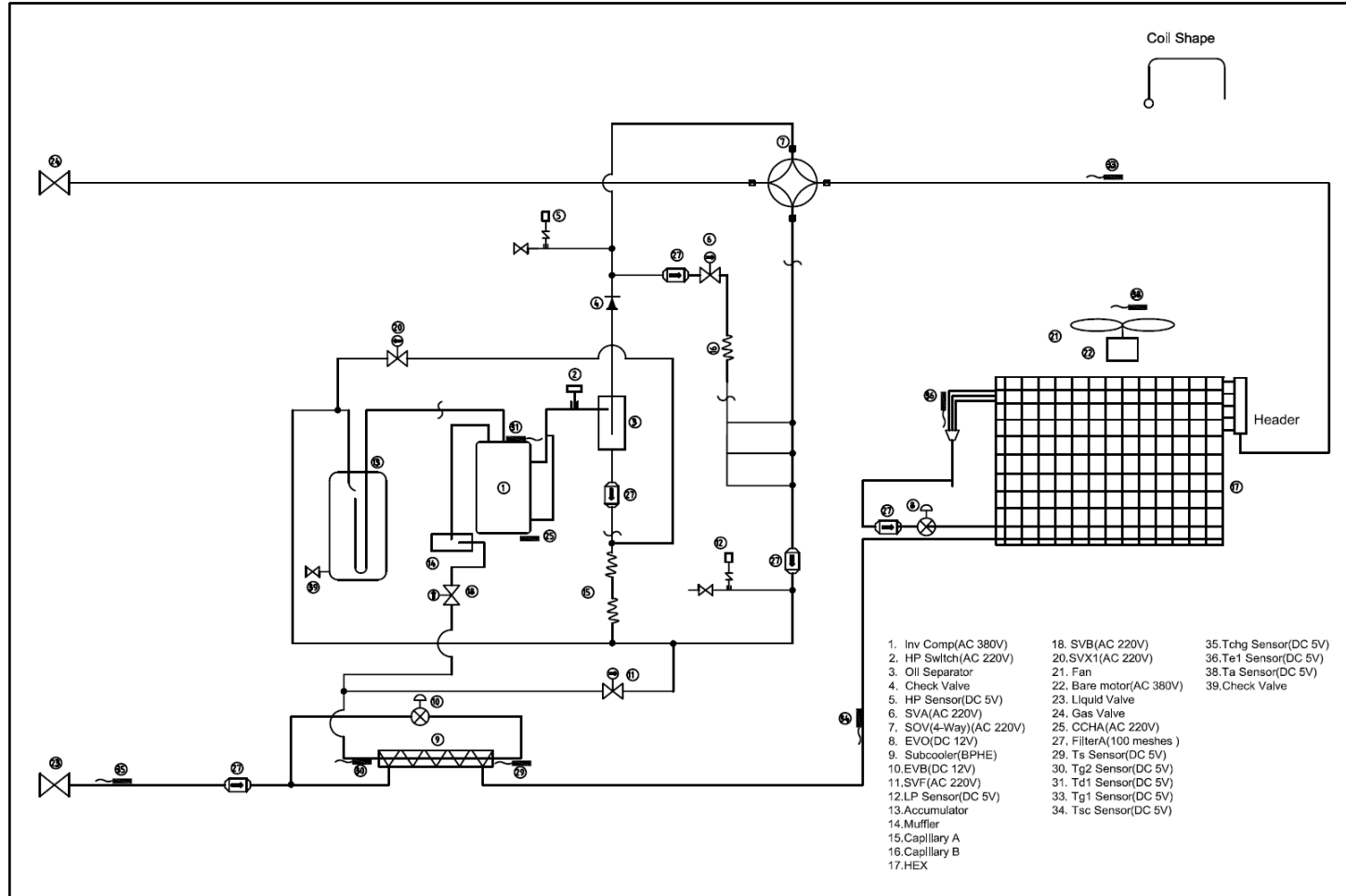
Item	Part Name	Item	Part Name
1	Compressor (Inverter) (2pcs)	15	Electrical Control Box
2	Heat Exchanger	16	Low Pressure Sensor
3	Propeller Fan	17	High Pressure Sensor
4	Fan Motor	18	High Pressure Switch for Protection (2 pcs)
5	Accumulator (Pressure Vessel)	19	Strainer
6	Oil Separator (Not Pressure Vessel) (2pcs)	20	Check Valve (2 pcs)
7	Micro-computer Controlled Expansion Valve (3 pcs)	21	Crankcase Heater (4 pcs)
8	4-Way Valve	22	Plate Heat Exchanger
9	Stop Valve (Gas)	23	Compressor Cover
10	Stop Valve (Liquid)	24	Terminal Block Box
11	Hot Gas Bypass Solenoid Valve (SVA)	25	Injection Muffler (2 pcs)
12	Check Joint (Low)	26	Injection Solenoid Valve (SVB/SVC)
13	Check Joint (High)	27	Injection Solenoid Valve (SVF)
14	Check Joint (For Oil)	28	Solenoid Valve (SVX1/SVX2)



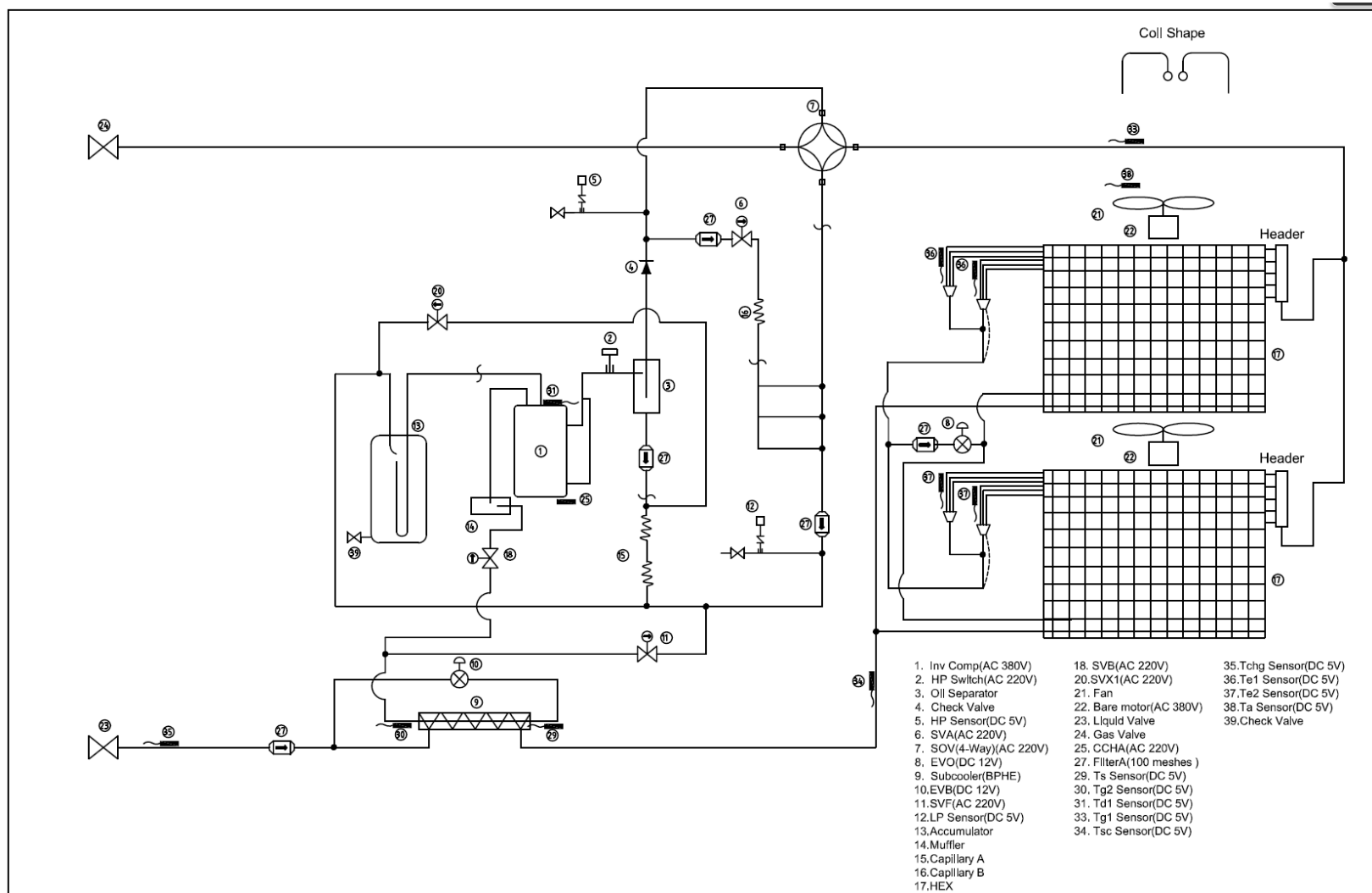
VRF Component

Refrigerant Cycle

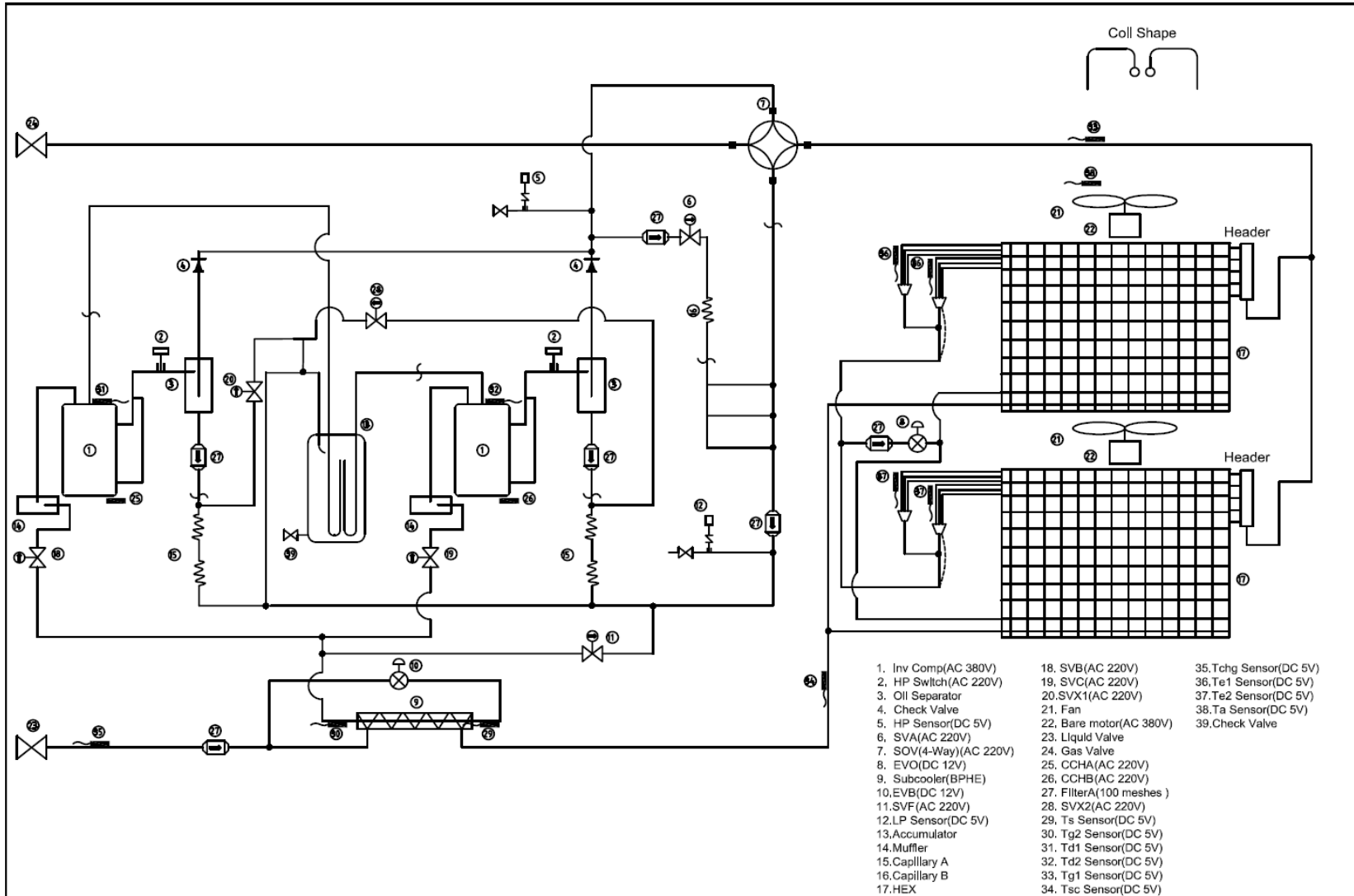
■ RAS-8.0HNBCM, RAS-10HNBCM, RAS-12HNBCM



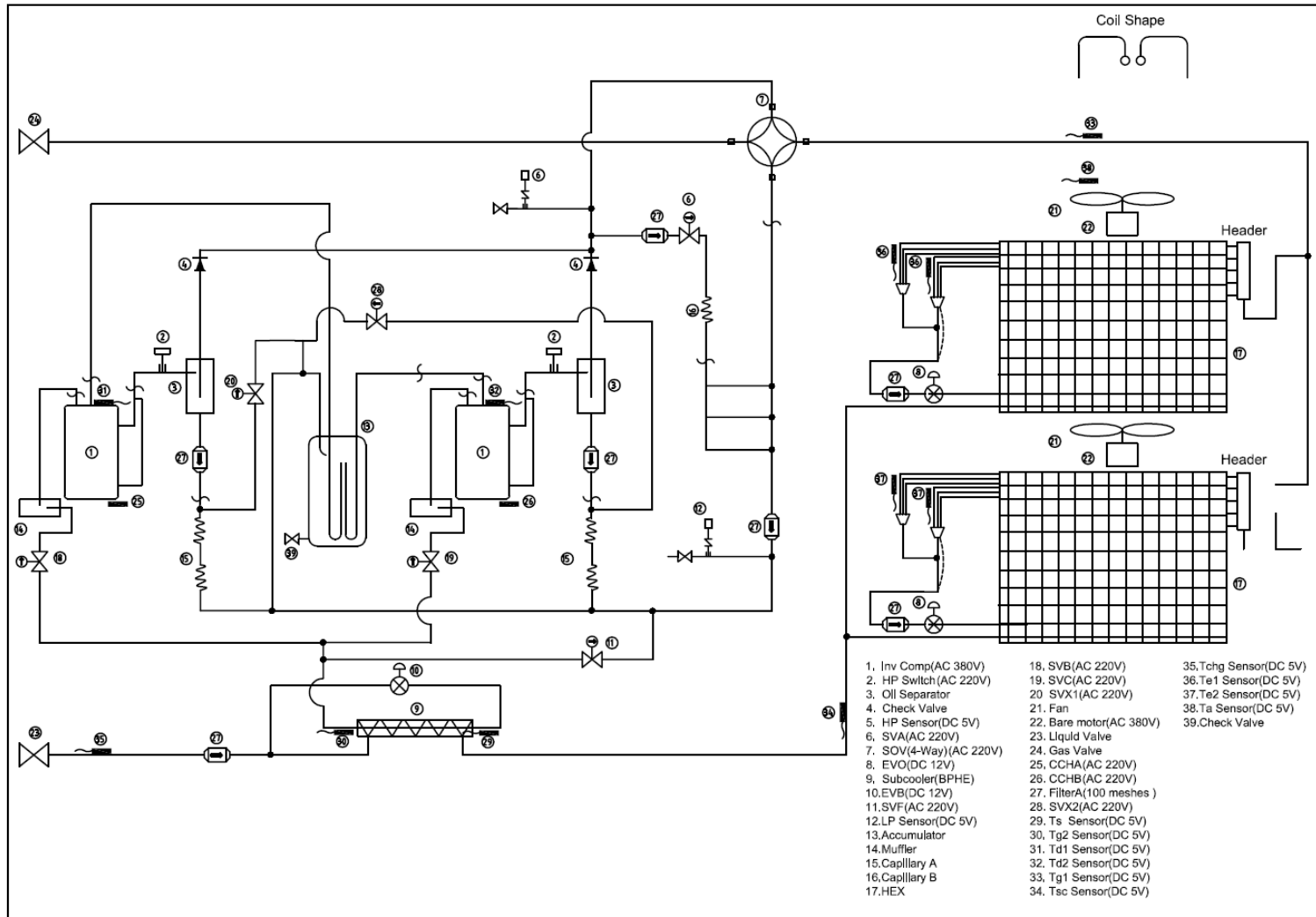
■ RAS-14HNBCM, RAS-16HNBCM



■ RAS-18HNBCMQR

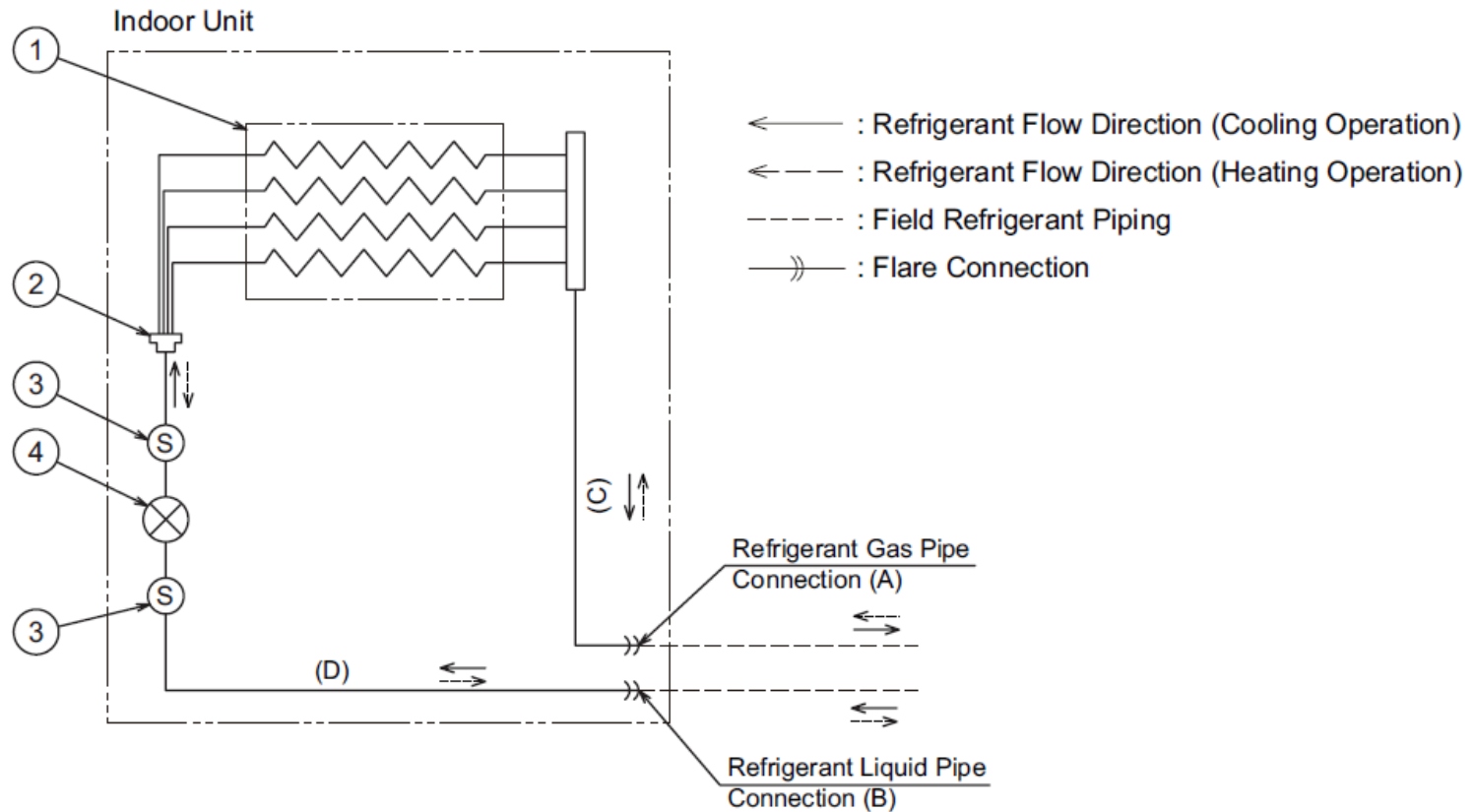


■ RAS-20HNBCMQRAS-22HNBCMQRAS-24HNBCMQR



■ Indoor unit

- Strainers before and after the EEV
- All indoor units (same configuration)





VRF Component

Compressor and Control

■ Compressor

Type	Outdoor Unit Models	Inverter Compressor		
		380-415V/50Hz	AA50PHDG-D1Y2	DC80PHDG-D1Y2
Standard Type	RAS-8.0HNBCM		1	
	RAS-10HNBCM		1	
	RAS-12HNBCM			1
	RAS-14HNBCM			1
	RAS-16HNBCM			1
	RAS-18HNBCM		2	
	RAS-20HNBCM		2	
	RAS-22HNBCM			2
	RAS-24HNBCM			2

■ Compressor(AA50)

1. 规格 Specification

1.1 压缩机 Compressor

压缩机型号 Compressor Model	AA50PHDG-D1Y2
压缩机型式 Compressor Type	直流变频压缩机 DC Compressor
压缩机方式 Compression Type	涡旋式 Scroll Type
使用冷媒 Refrigerant	R410A
排气容积 Displacement	50cm ³ /rev
润滑油 / 油量 Oil / Oil Charge	FV68H / 1100ml
涂装 Painting	黑色 Black Color Paint
重量(含油) Net Mass(Including Oil)	33kg
吸气管接口内径 Suction Accept I.D	Φ 22.4mm
排气管接口内径 Discharge Accept I.D	Φ 16.1mm
喷气管接口内径 Injection Accept I.D	Φ 9.7mm

1.2 电机 Motor

变频器电源 Inverter Power	380-415V, 50/60Hz
电动机型式 Motor Type / 起动方式 Star Mode	三相直流变频同步电机 / 直流专用变频器起动 3 Phase DC Synchronous Motor / DC Inverter Starting
极数 Poles	6极 6 Poles
运转频率范围 Running Frequency Range	45~420Hz
转速范围 Rotate Speed Range	15~140rps
绝缘等级 Insulation Class	E级 E Class
绕组阻抗 Winding Resistance(at 20°C)	0.197±7%Ω

1.3 性能 Performance

项目 Items	转速 Rotate Speed	规格值 Nominals	允差 Franchise
制冷能力 Capacity	At 90rps (ARI 工况)	24660W	95% Min
输入功率 Motor Input		8060W	105% Max
能效比 EER		3.06W/W	95% Min
噪音 Sound Level		75dB (A) Max	—
振动 Vibration		100μm Max	—

ARI 测试条件 Test condition of ARI:

冷凝温度 Condensing temp.: 54.4 °C

回气温度 Return gas temp.: 18.3 °C

膨胀阀前液体温度 Liquid temp.: 46.0 °C

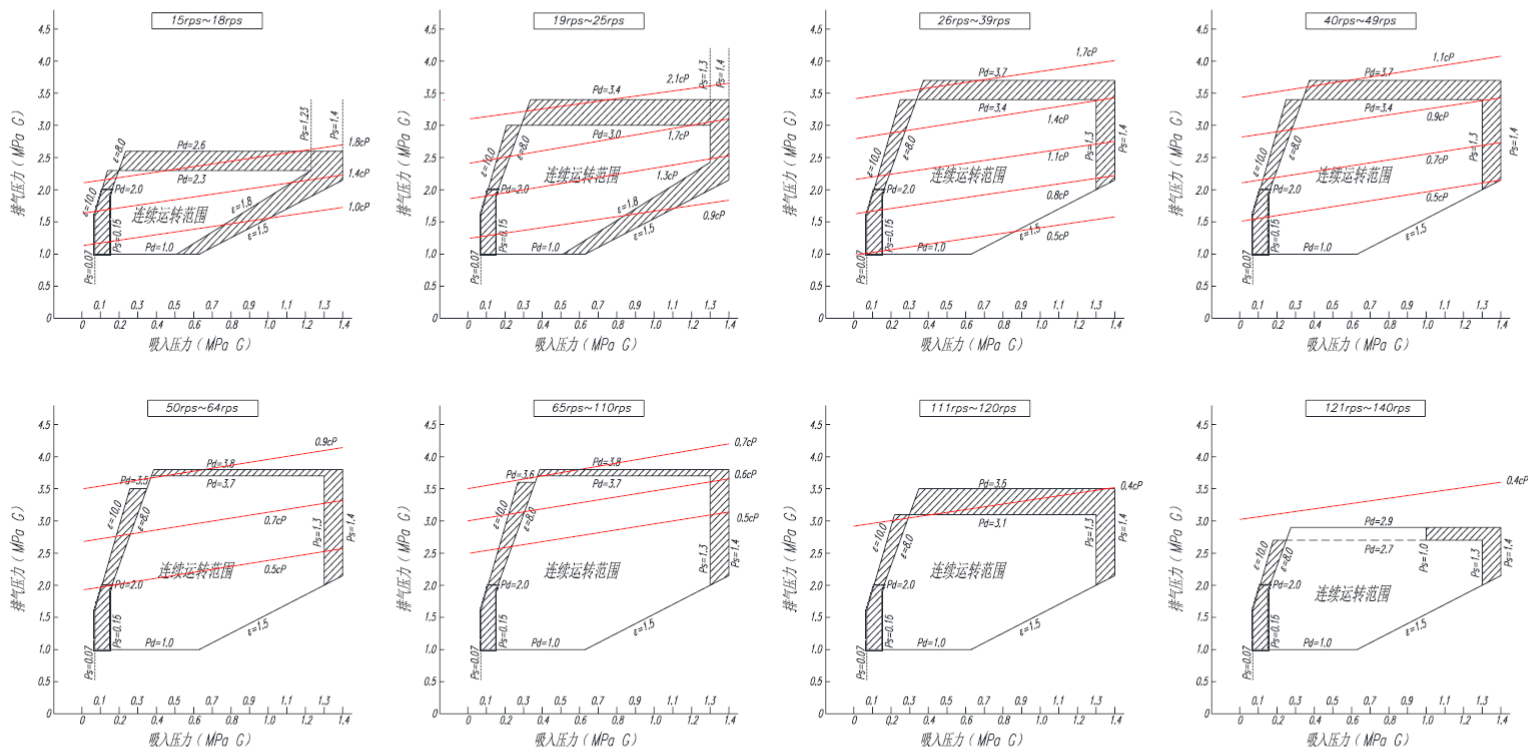
蒸发温度 Evaporating temp.: 7.2 °C

周围温度 Ambient temp.: 35.0 °C

Compressor(AA50)

压缩机运行范围 (Running Range Of Compressor)

适用机型	AA50PHD
使用冷媒	R410A
润滑油	FV68H



- 注: 1. 斜线内是带条件的运行压力范围。
 2. 其中带条件的运行范围运行时间累计不超过500小时。
 3. 确保图示的混合润滑油粘度以上。
 (例) 50rps Pd=3.7MPa, 确保粘度0.7cP以上
 4. 转速在40rps以上60rps以下时,
 以下条件范围内混合润滑油粘度不能确保时也可以运行:
 压力条件: 排气压力在2.7MPa以下连续运行。
 时间与频率: 13分/次以下, 1次/天。 油粘度: 0.3cP以上

5. 双点划线区域内排气温度应在90℃以下:
 ①过渡阶段(3分钟内)在该区域运转无累计时间限制;
 ②超过3分钟累计在500h内。
 6. 以上压力均为表压。

制图				型号:	编号:
校对			广州日立压缩机有限公司	压缩机运行范围图	EL-18
审核					

■ Compressor(DC80)

1. 规格 Specification

1.1 压缩机 Compressor

压缩机型号 Compressor Model	DC80PHDG-D1Y2
压缩机型式 Compressor Type	直流变频压缩机 DC Compressor
压缩机方式 Compression Type	涡旋式 Scroll Type
使用冷媒 Refrigerant	R410A
排气容积 Displacement	80cm ³ /rev
润滑油 / 油量 Oil / Oil Charge	FV68H / 1100ml
涂装 Painting	黑色 Black Color Paint
重量(含油) Net Mass(Including Oil)	39kg
吸气管接口内径 Suction Accept I.D	Φ22.4mm
排气管接口内径 Discharge Accept I.D	Φ16.1mm
喷气管接口内径 Injection Accept I.D	Φ9.7mm

1.2 电机 Motor

变频器电源 Inverter Power	380-415V, 50/60Hz
电动机型式 Motor Type / 启动方式 Star Mode	三相直流变频同步电机 / 直流专用变频器启动 3 Phase DC Synchronous Motor / DC Inverter Starting
极数 Poles	6极 6 Poles
运转频率范围 Running Frequency Range	30~390Hz
转速范围 Rotate Speed Range	10~130rps
绝缘等级 Insulation Class	E级 E Class
绕组阻抗 Winding Resistance (at 20°C)	0.124±7%Ω

1.3 性能 Performance

项目 Items	规格值 Nominals	参考值 Reference	允差 Franchise
转速 Rotate Speed	60rps (ARI 工况)	90rps (ARI 工况)	
制冷能力 Capacity	26400W	39500W	95% Min
输入功率 Motor Input	8130W	12910W	105% Max
能效比 EER	3.25W/W	3.06W/W	95% Min
噪音 Sound Level	72dB (A) Max	—	—
振动 Vibration	100μm Max	—	—

ARI 测试条件 Test condition of ARI:

冷凝温度 Condensing temp.: 54.4 °C

回气温度 Return gas temp.: 18.3 °C

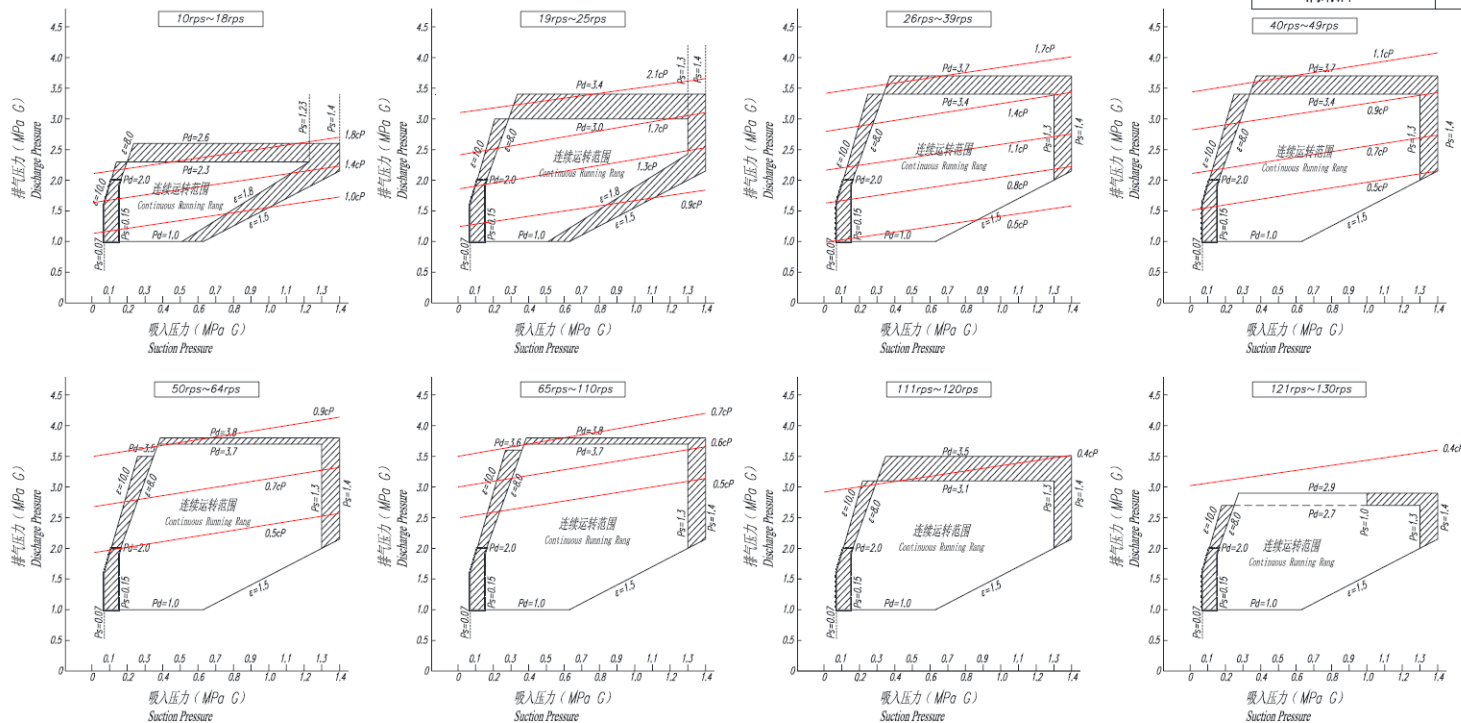
膨胀阀前液体温度 Liquid temp.: 46.1 °C

蒸发温度 Evaporating temp.: 7.2 °C

周围温度 Ambient temp.: 35.0 °C

Compressor

压缩机运行范围 (Running Range Of Compressor)



适用机型 Model	DC65PHDG/DC80PHDG
使用冷媒 Refrigerator	R410A
润滑油 Oil	FV68H

- 注: 1. 内是带条件的运行压力范围, The operating pressure range with conditions
- 其中等条件的运行范围运行时间累计不超过500小时, With conditions of operation range of the time not more than 500 hours
 - 确保图示的混合润滑油粘度以上。 Make sure the viscosity of mixed lubrication is higher than the graphical curve.
(例) 50rps $P_d=3.7\text{MPa}$, 确保粘度0.7cP以上 (Example) $P_d=3.7\text{MPa}$ 50rps, to ensure that the viscosity of 0.7cP above
 - 转速在40rps以上60rps以下时, 以下条件范围内混合润滑油粘度不能确保时也可以运行:
When speed is between 40rps and 60rps, compressor can also operate although viscosity of mixed lubrication could not ensure under the following conditions;
压力条件: 排气压力在2.7MPa以下连续运行。 Pressure conditions: exhaust pressure in the following continuous operation of 2.7MPa.
时间与频率: 13分/次以下, 1次/天。 Time and frequency: less than 13 min / time, 1 times / day.
油粘度: 0.3cP以上。 Oil viscosity: 0.3cP above.

5. 双点划线区域内排气温度应在90℃以下: The exhaust temperature in the double dot line area should be below 90;
①过渡阶段(3分钟内)在该区域运转无累计时间限制; The transition phase (3 minutes) in the area of operation without the cumulative time limit;
②超过3分钟累计在500h内。 More than 3 minutes accumulated in the 500h.
6. 以上压力均为表压。 The graphic pressure for relative pressure.

制图				
校对			广州日立压缩机有限公司	
审核				

型号: 压缩机运行范围图
编号: EL-22

■ Compressor control

■ Compressor Capacity Control

The operating speed of the compressor is determined according to the temperature difference (ΔT) between setting temperature and indoor unit air inlet temperature detected by each indoor unit under cooling/heating thermo-ON operation and the variation of ΔT to control compressor frequency.

The frequency is calculated as follows:

Current Frequency \times Coefficient Based on the Temperature

(for Cooling Operation)

The coefficient becomes larger when the value of ΔT (the temperature difference between setting temperature and air inlet temperature is large) or variation of ΔT is larger.

The coefficient becomes smaller when the value of ΔT (the temperature difference between setting temperature and air inlet temperature is small) or variation of ΔT is smaller.

(for Heating Operation)

The coefficient becomes larger when the value of ΔT (the temperature difference between setting temperature and air inlet temperature is large) or variation of ΔT is larger.

The coefficient becomes smaller when the value of ΔT (the temperature difference between setting temperature and air inlet temperature is small) or variation of ΔT is smaller.

$$F_c(n) = F_c(n-1) \times \Delta F$$

$F_c(n)$: n frequency

$F_c(n-1)$: n-1 frequency

ΔF : Coefficient Base on the temperature

Compressor control

Compressor Capacity Control Cooling

$$F_c(n) = F_c(n-1) \times \Delta F$$

$F_c(n)$: n frequency

$F_c(n-1)$: n-1 frequency

ΔF : Coefficient Base on the temperature

$$Q_{\Delta T1c} : \Delta T1c(n) - \Delta T1c(n-1)$$

$\Delta T1c(n) : T_i - T_s$

$\Delta T1c(n-1) : n-1$ difference value

$\Delta F : \max(\text{indoor units } \Delta Hzc)$



ΔHzc	$\Delta T1c(n) < -2$	$-2 \leq \Delta T1c(n) < -1$	$-1 \leq \Delta T1c(n) < 0$	$0 \leq \Delta T1c(n) < 1$	$1 \leq \Delta T1c(n) < 2$	$2 \leq \Delta T1c(n) < 3$	$3 \leq \Delta T1c(n) < 5$	$5 \leq \Delta T1c(n) < 7$	$7 \leq \Delta T1c(n)$
$4 < Q_{\Delta T1c}$	1.14	1.16	1.18	1.20	1.21	1.22	1.23	1.25	1.30
$2 < Q_{\Delta T1c} \leq 4$	1.09	1.11	1.13	1.15	1.16	1.17	1.18	1.20	1.25
$1 < Q_{\Delta T1c} \leq 2$	1.04	1.06	1.08	1.10	1.11	1.12	1.13	1.15	1.20
$0 < Q_{\Delta T1c} \leq 1$	0.99	1.01	1.03	1.05	1.06	1.07	1.08	1.10	1.15
$-1 < Q_{\Delta T1c} \leq 0$	0.85	0.90	0.95	1.00	1.01	1.02	1.03	1.05	1.10
$-2 < Q_{\Delta T1c} \leq -1$	0.80	0.85	0.90	0.95	0.96	0.97	0.98	1.00	1.05
$-3 < Q_{\Delta T1c} \leq -2$	0.75	0.80	0.85	0.90	0.91	0.92	0.93	0.95	1.00
$-5 < Q_{\Delta T1c} \leq -3$	0.70	0.75	0.80	0.85	0.86	0.87	0.88	0.90	0.95
$Q_{\Delta T1c} \leq -5$	0.65	0.70	0.75	0.80	0.81	0.82	0.83	0.85	0.90

■ Compressor control

■ Compressor Capacity Control Heating

$$F_c(n) = F_c(n-1) \times \Delta F$$

$F_c(n)$: n frequency

$F_c(n-1)$: n-1 frequency

ΔF : Coefficient Base on the temperature

$$Q_{\Delta T1h} : \Delta T1h(n) - \Delta T1h(n-1)$$

$\Delta T1h(n)$: n, $T_s - T_i$, difference value of setting temperature and indoor unit inlet temperature

$\Delta T1h(n-1)$: n-1, difference value

ΔF : max (indoor units ΔHz)

indoor

← Inlet tem: T_i

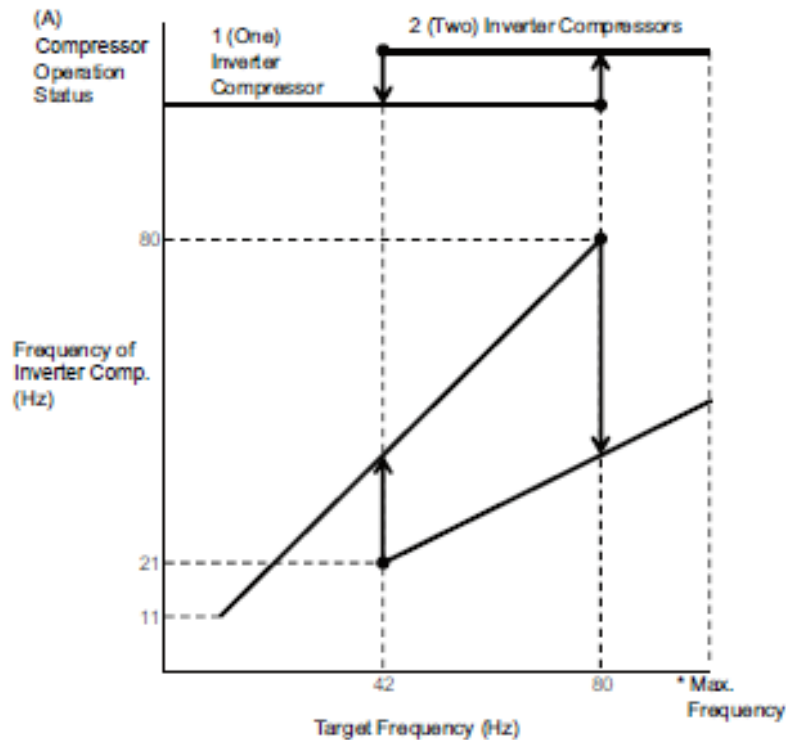


T_s

ΔHz	$\Delta T1h(n) < -2$	$-2 \leq \Delta T1h(n) < -1$	$-1 \leq \Delta T1h(n) < 0$	$0 \leq \Delta T1h(n) < 1$	$1 \leq \Delta T1h(n) < 2$	$2 \leq \Delta T1h(n) < 3$	$3 \leq \Delta T1h(n) < 5$	$5 \leq \Delta T1h(n) < 7$	$7 \leq \Delta T1h(n)$
$4 < Q_{\Delta T1h}$	1.14	1.16	1.18	1.20	1.21	1.22	1.23	1.25	1.30
$2 < Q_{\Delta T1h} \leq 4$	1.09	1.11	1.13	1.15	1.16	1.17	1.18	1.20	1.25
$1 < Q_{\Delta T1h} \leq 2$	1.04	1.06	1.08	1.10	1.11	1.12	1.13	1.15	1.20
$0 < Q_{\Delta T1h} \leq 1$	0.99	1.01	1.03	1.05	1.06	1.07	1.08	1.10	1.15
$-1 < Q_{\Delta T1h} \leq 0$	0.85	0.90	0.95	1.00	1.01	1.02	1.03	1.05	1.10
$-2 < Q_{\Delta T1h} \leq -1$	0.80	0.85	0.90	0.95	0.96	0.97	0.98	1.00	1.05
$-3 < Q_{\Delta T1h} \leq -2$	0.75	0.80	0.85	0.90	0.91	0.92	0.93	0.95	1.00
$-5 < Q_{\Delta T1h} \leq -3$	0.70	0.75	0.80	0.85	0.86	0.87	0.88	0.90	0.95
$Q_{\Delta T1h} \leq -5$	0.65	0.70	0.75	0.80	0.81	0.82	0.83	0.85	0.90

■ RAS-18HNBCM to 20HNBCM(Compressor Stepping)

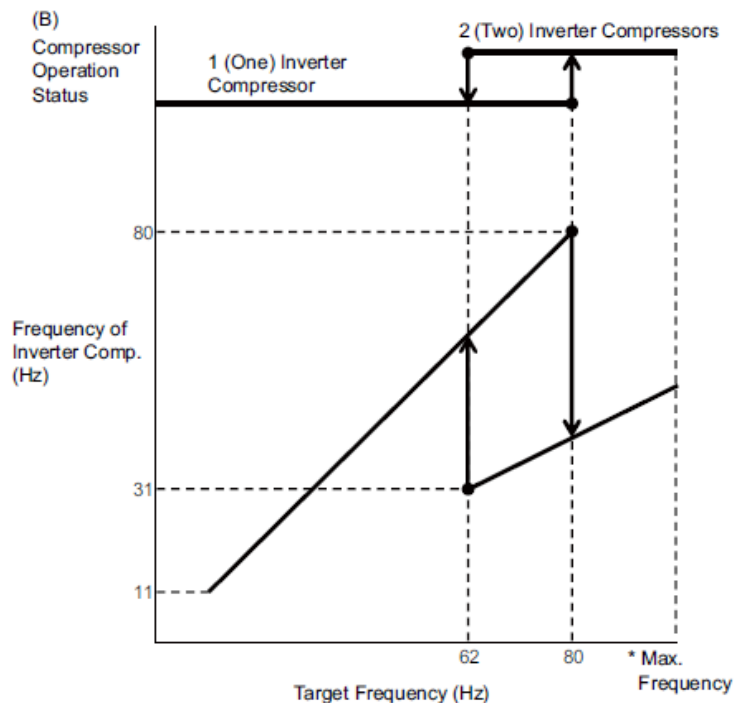
- 2 Inverter compressors
- Compressor Operation Control is to adjust the output frequency of an Inverter Compressor according to Target Frequency



Target Frequency [Hz]	Each Compressor Frequency [Hz]					
	Increase Direction			Decrease Direction		
	Comp. Operation Status	No.1 Comp.	No.2 Comp.	Comp. Operation Status	No.1 Comp.	No.2 Comp.
11.0	1	11.0	—	1	11.0	—
41.0	1	41.0	—	1	41.0	—
42.0	1	42.0	—	2	21.0	21.0
80.0	1	80.0	—	2	40.0	40.0
81.0	2	40.5	40.5	2	40.5	40.5
•	•	•	•	•	•	•
•	•	•	•	•	•	•
•	•	•	•	•	•	•

■ RAS-22HNBCM to 24HNBCM(Compressor Stepping)

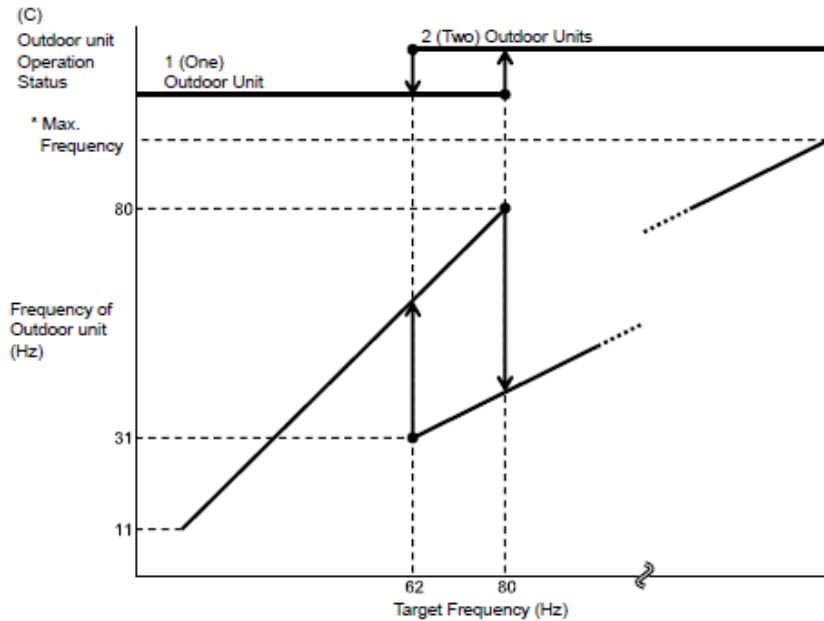
- 2 Inverter compressors
- Compressor Operation Control is to adjust the output frequency of an Inverter Compressor according to Target Frequency



Target Frequency [Hz]	Each Compressor Frequency [Hz]					
	Increase Direction			Decrease Direction		
	Comp. Operation Status	No.1 Comp.	No.2 Comp.	Comp. Operation Status	No.1 Comp.	No.2 Comp.
11.0	1	11.0	—	1	11.0	—
61.0	1	61.0	—	1	61.0	—
62.0	1	62.0	—	2	31.0	31.0
80.0	1	80.0	—	2	40.0	40.0
81.0	2	40.5	40.5	2	40.5	40.5
•	•	•	•	•	•	•
•	•	•	•	•	•	•
•	•	•	•	•	•	•

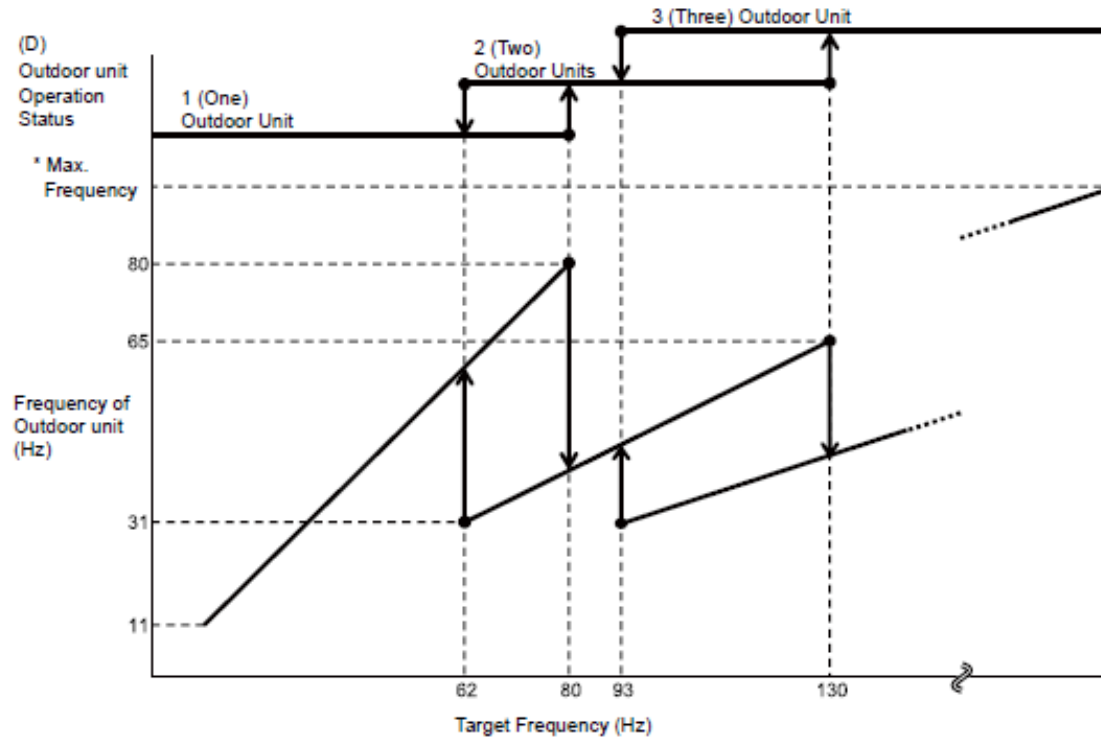
■ RAS-26HNBCM to 48HNBCM (Compressor Stepping)

■ Multiple outdoor modules



Target Frequency [Hz]	Each Outdoor Unit Frequency [Hz]					
	Increase Direction			Decrease Direction		
	Outdoor unit Operation Status	Outdoor Unit (A)	Outdoor Unit (B)	Outdoor unit Operation Status	Outdoor Unit (A)	Outdoor Unit (B)
11.0	1	11.0	—	1	11.0	—
61.0	1	61.0	—	1	61.0	—
62.0	1	62.0	—	2	31.0	31.0
80.0	1	80.0	—	2	40.0	40.0
81.0	2	40.5	40.5	2	40.5	40.5
⋮	⋮	⋮	⋮	⋮	⋮	⋮
⋮	⋮	⋮	⋮	⋮	⋮	⋮
⋮	⋮	⋮	⋮	⋮	⋮	⋮

- RAS-50HNBCM to 72HNBCM (Compressor Stepping)
- Multiple outdoor modules

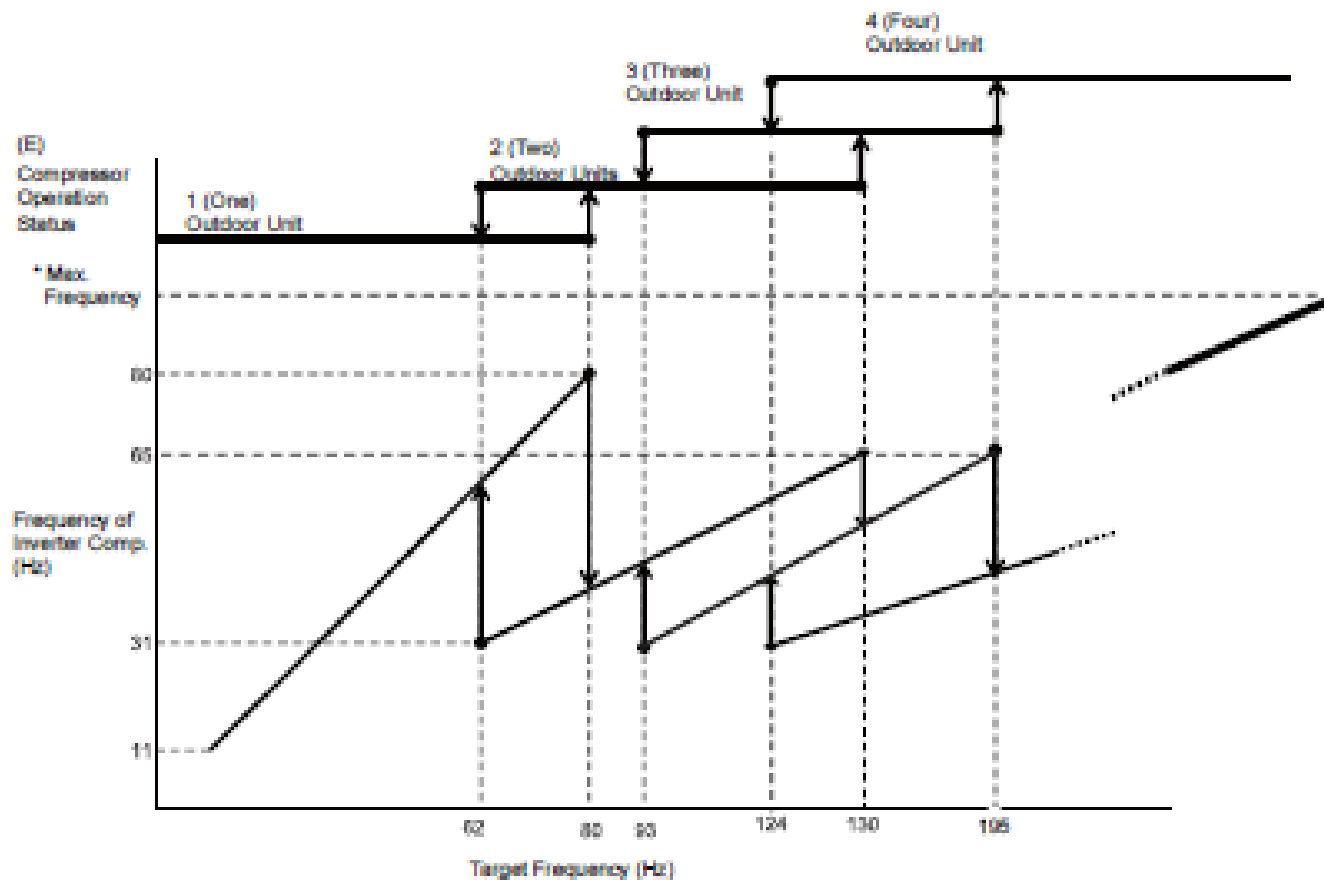


■ Compressor Stepping (Cont'd.)

Target Frequency [Hz]	Each Outdoor Unit Frequency [Hz]							
	Increase Direction				Decrease Direction			
	Outdoor unit Operation Status	Outdoor Unit (A)	Outdoor Unit (B)	Outdoor Unit (C)	Outdoor unit Operation Status	Outdoor Unit (A)	Outdoor Unit (B)	Outdoor Unit (C)
11.0	1	11.0	—	—	1	11.0	—	—
61.0	1	61.0	—	—	1	61.0	—	—
62.0	1	62.0	—	—	2	31.0	31.0	—
80.0	1	80.0	—	—	2	40.0	40.0	—
81.0	2	40.5	40.5	—	2	40.5	40.5	—
92.0	2	46.0	46.0	—	2	46.0	46.0	—
93.0	2	46.5	46.5	—	3	31.0	31.0	31.0
130.0	2	65.0	65.0	—	3	43.3	43.3	43.3
131.0	3	43.6	43.6	43.6	3	43.6	43.6	43.6
⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮
⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮
⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮

■ Compressor Stepping (Cont'd.)

- Multiple outdoor modules
- Similar operation as a single outdoor module system



■ Compressor Stepping (Cont'd.)

Target Frequency [Hz]	Each Outdoor Unit Frequency [Hz]									
	Increase Direction					Decrease Direction				
	Outdoor unit Operation Status	Outdoor Unit (A)	Outdoor Unit (B)	Outdoor Unit (C)	Outdoor Unit (D)	Outdoor unit Operation Status	Outdoor Unit (A)	Outdoor Unit (B)	Outdoor Unit (C)	Outdoor Unit (D)
11.0	1	11.0				1	11.0			
61.0	1	61.0				1	61.0			
62.0	1	62.0				2	31.0	31.0		
80.0	1	80.0				2	40.0	40.0		
81.0	2	40.5	40.5			2	40.5	40.5		
92.0	2	46.0	46.0			2	46.0	46.0		
93.0	2	46.5	46.5			3	31.0	31.0	31.0	
123.0	2	61.5	61.5			3	41.0	41.0	41.0	
124.0	2	62.0	62.0			4	31.0	31.0	31.0	31.0
130.0	2	65.0	65.0			4	32.5	32.5	32.5	32.5
131.0	3	43.6	43.6	43.6		4	32.8	32.8	32.8	32.8
195.0	3	65.0	65.0	65.0		4	48.8	48.8	48.8	48.8
196.0	4	49.0	49.0	49.0	49.0	4	49.0	49.0	49.0	49.0
⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮
⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮	⋮

■ Maximum Frequency

Model	Combination of Base Units	Maximum Frequency [Hz]
RAS-26HNBCM-Q	10	90
	16	148
RAS-28HNBCM-Q	12	110
	16	148
RAS-30HNBCM-Q	14	130
	16	148
RAS-32HNBCM-Q	16	148
	16	148
RAS-34HNBCM-Q	16	148
	18	200
RAS-36HNBCM-Q	16	148
	20	200
RAS-38HNBCM-Q	16	148
	22	250
RAS-40HNBCM-Q	16	148
	24	260
RAS-42HNBCM-Q	18	200
	24	260
RAS-44HNBCM-Q	20	200
	24	260
RAS-46HNBCM-Q	22	250
	24	260
RAS-48HNBCM-Q	24	260
	24	260

RAS-50HNBCM-Q	16	148
	16	148
	18	200
RAS-52HNBCM-Q	16	148
	16	148
	20	200
RAS-54HNBCM-Q	16	148
	16	148
	22	250
RAS-56HNBCM-Q	16	148
	16	148
	24	260
RAS-58HNBCM-Q	16	148
	18	200
	24	260
RAS-60HNBCM-Q	16	148
	20	200
	24	260
RAS-62HNBCM-Q	16	148
	22	250
	24	260
RAS-64HNBCM-Q	16	148
	24	260
	24	260
RAS-66HNBCM-Q	18	200
	24	260
	24	260
RAS-68HNBCM-Q	20	200
	24	260
	24	260
RAS-70HNBCM-Q	22	250
	24	260
	24	260

■ Maximum Frequency

Model	Combination of Base Units	Maximum Frequency [Hz]
RAS-72HNBCM-Q	24	260
	24	260
	24	260
RAS-74HNBCM-Q	16	148
	16	148
	18	200
RAS-76HNBCM-Q	24	260
	16	148
	16	148
RAS-78HNBCM-Q	20	200
	24	260
	16	148
RAS-80HNBCM-Q	16	148
	22	250
	24	260
RAS-82HNBCM-Q	16	148
	20	200
	20	200
	20	200

Model	Combination of Base Units	Maximum Frequency [Hz]
RAS-84HNBCM-Q	20	200
	20	200
	20	200
	24	260
RAS-86HNBCM-Q	20	200
	20	200
	22	250
	24	260
RAS-88HNBCM-Q	20	200
	20	200
	24	260
	24	260
RAS-90HNBCM-Q	20	200
	22	250
	24	260
	24	260
RAS-92HNBCM-Q	20	200
	24	260
	24	260
	24	260
RAS-94HNBCM-Q	22	250
	24	260
	24	260
	24	260
RAS-96HNBCM-Q	24	260
	24	260
	24	260
	24	260

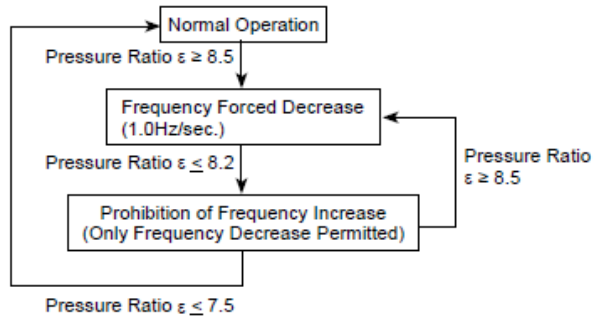
■ Compressor control

■ Compressor Protection Control

(a) Pressure Ratio Increase Protection Control

Pressure Ratio Increase Protection Control is performed in order to protect the compressor from an increase of pressure ratio.

<Details of Control>



NOTE:

1. With a combination of base units, the control in the figure is performed for the entire number of outdoor units to be connected.
2. The pressure ratio is calculated in each outdoor unit, and this control uses the maximum value.

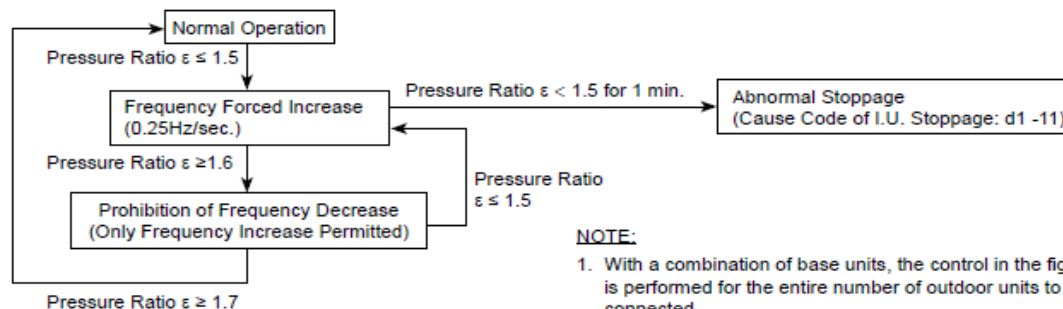
$$\epsilon = (Pd \text{ [MPa]} + 0.1) / (Ps \text{ [MPa]} + 0.06)$$

$$\epsilon = (Pd \text{ [psi]} + 15) / (Ps \text{ [psi]} + 9)$$
 Pd: Detected Value of High Pressure Sensor [MPa (psi)]
 Ps: Detected Value of Low Pressure Sensor [MPa (psi)]

(b) Low Compression Ratio Protection Function

This function is activated to protect the compressor during occurrences of low compression ratio.

<Details of Control>



NOTE:

1. With a combination of base units, the control in the figure is performed for the entire number of outdoor units to be connected.
2. The pressure ratio is calculated in each outdoor unit, and this control uses the minimum value.

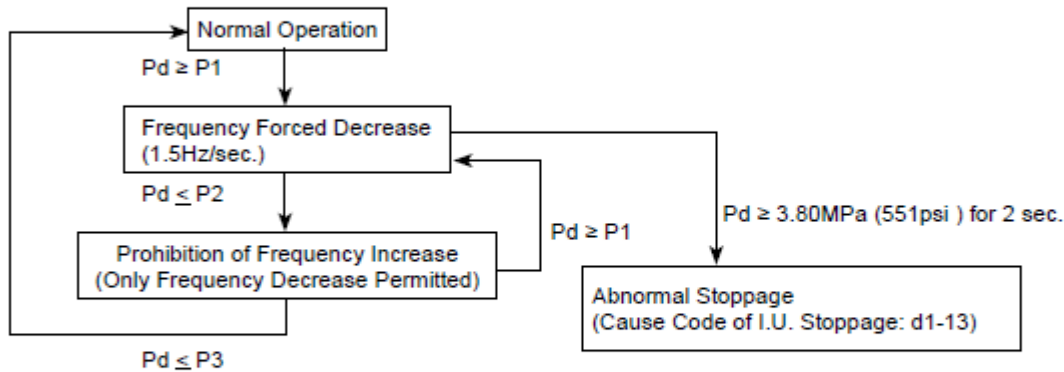
■ Compressor control

■ Compressor Protection Control

P02: High Pressure Increase Protection Control

High Pressure Protection Control is performed in order to prevent activation of a protection device caused by a high pressure increase during an abnormality and to protect the compressor from an excessive increase of discharge pressure.

<Details of Control>



Control Value [MPa(psi)]

Operation Mode	P1	P2	P3
Cooling	3.45 (500)	3.40 (493)	3.20 (464)
Heating	3.35 (486)	3.30 (479)	3.10 (450)

NOTE:

1. With a combination of base units, the control in the figure is performed for the entire number of outdoor units to be connected.
2. High pressure is detected in each outdoor unit, and this control uses the maximum value.

Pd: Detected Value of High Pressure Sensor [MPa(pis)]

■ Compressor control

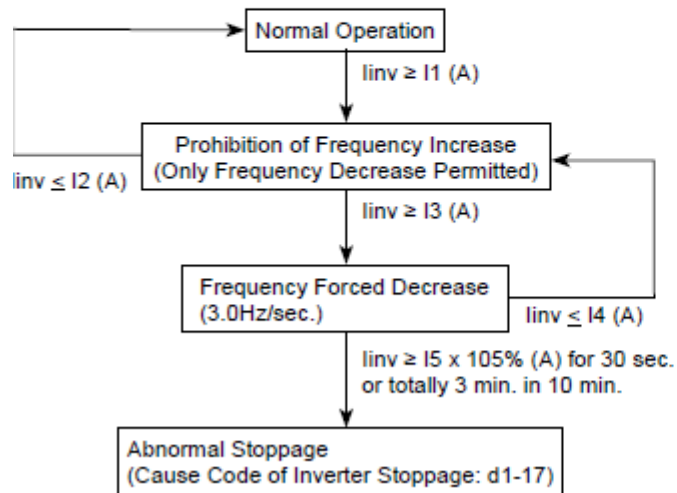
■ Compressor Protection Control

▮ P03: Inverter Current Protection Control

Inverter Current Protection Control is performed in order to prevent an inverter trip caused by an increase of inverter secondary current value.

(a) Inverter Secondary Current Protection

<Details of Control>



Control Value

<380-415V>

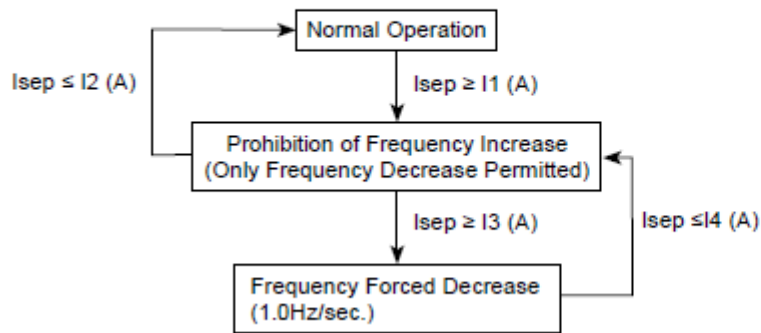
Model	I1	I2	I3	I4	I5
RAS-8.0HNBCM RAS-10HNBCM RAS-18HNBCM RAS-20HNBCM	22.8	22.3	23.8	23.3	23.8
RAS-12HNBCM RAS-14HNBCM RAS-16HNBCM RAS-22HNBCM RAS-24HNBCM	36.2	35.7	37.2	36.7	37.2

■ Compressor control

■ Compressor Protection Control

(b) Primary Current Protection for each Inverter PCB

<Details of Control>



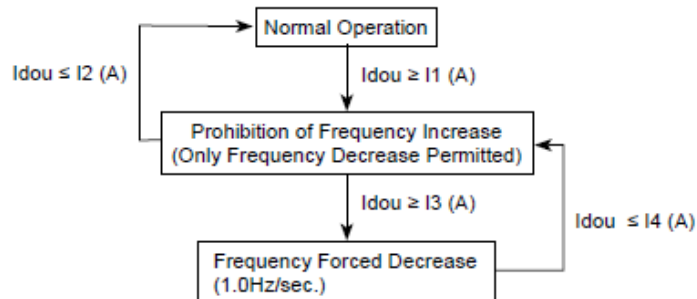
Control Value

<380-415V>

Model	I1	I2	I3	I4
RAS-8.0HNBCMQRAS-10HNBCMQRAS-18HNBCMQRAS-20HNBCMQR	24.7	23.7	25.2	24.7
RAS-12HNBCMQRAS-14HNBCMQRAS-16HNBCMQRAS-22HNBCMQRAS-24HNBCMQR	39.1	38.1	39.6	39.1

(c) Primary Current Protection for each Outdoor Unit

<Details of Control>



Control Value

<380-415V>

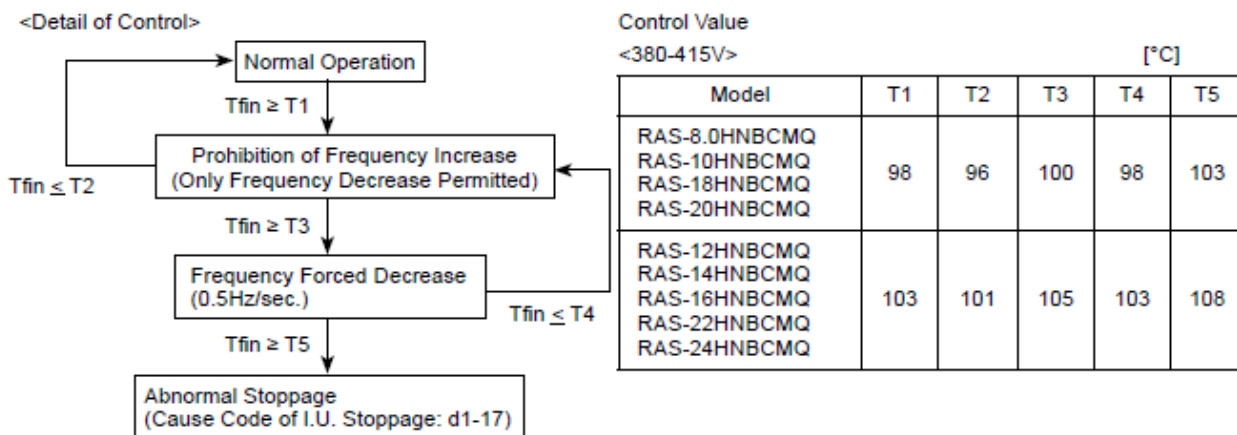
Model	I1	I2	I3	I4
RAS-8.0HNBCMQRAS-10HNBCMQRAS-12HNBCMQRAS-14HNBCMQRAS-16HNBCMQRAS-18HNBCMQRAS-20HNBCMQRAS-22HNBCMQRAS-24HNBCMQR	14.8	13.8	15.3	14.8
	20.4	19.4	20.9	20.4
	24.4	23.4	24.9	24.4
	28.6	27.6	29.1	28.6
	32.2	31.2	32.7	32.2
	41.6	40.6	42.1	41.6
	41.6	40.6	42.1	41.6
	48.2	47.2	48.7	48.2
	57.0	56.0	57.5	57.0

■ Compressor control

■ Compressor Protection Control

P04: Inverter Fin Temperature Increase Protection Control

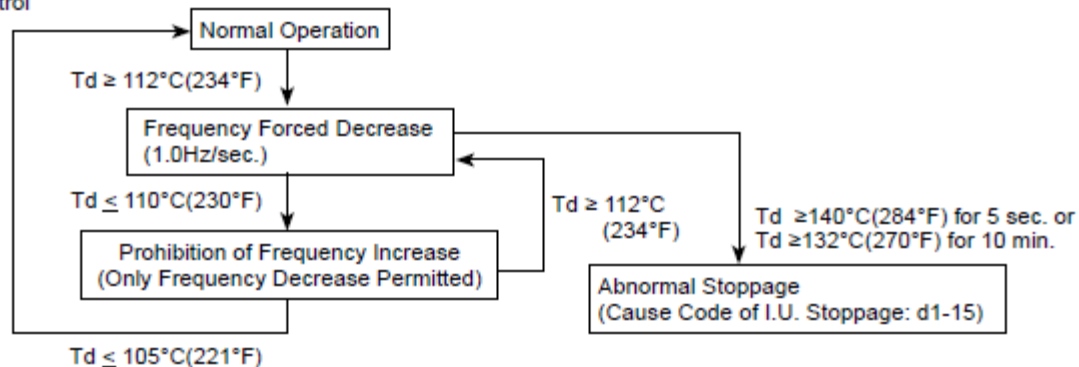
Inverter Fin Temperature Increase Protection Control is performed in order to prevent an inverter trip caused by a temperature increase of the inverter fin.



P05: Discharge Temperature Increase Protection Control

Discharge Temperature Increase Protection Control is performed in order to protect the compressor motor coil from an increase of discharge temperature during an abnormality.

Details of Control



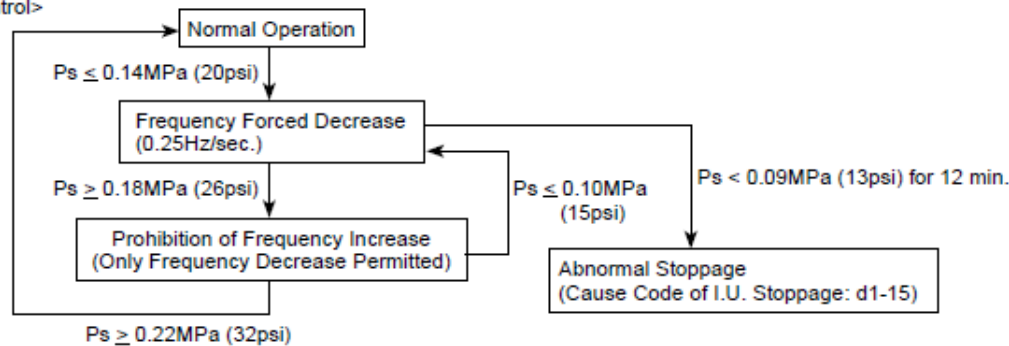
■ Compressor control

■ Compressor Protection Control

P06: Low Pressure Decrease Protection Control

Low Pressure Decrease Protection Control is performed in order to protect the compressor from a transitional decrease of suction pressure.

<Details of Control>

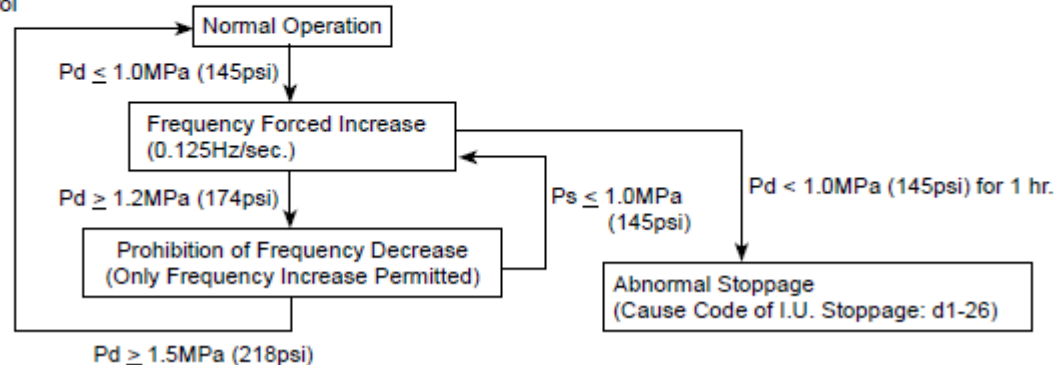


P09: High Pressure Decrease Protection Control

When decreasing high pressure, the compressor operation frequency is controlled by this protection control for the following purposes.

- To prevent insufficient refrigerant supply to indoor units installed at different height locations.
- To keep the refrigerant oil supply in the compressor.

Details of Control



■ Compressor control

■ Oil return Control

2) Oil Return Control

Oil return control is performed in order to avoid insufficient oil supply to the compressor caused by long time low frequency operation. This control is utilized to return the oil flow out to the indoor unit side from the compressor.

<Activating Condition>

This control function is started the compressor runs below the specified speed for 1 hour continuously (refer to the table below).

<Compressor Speed for Oil Return Control>

Type of Unit	Cooling Operation	Heating Operation
≤ 8HP	32.0	32.0
10HP	38.0	38.0
12HP, 14HP	50.0	54.0
16HP to 24HP	66.0	72.0
26HP to 36HP	96.0	108.0
38HP to 54HP	132.0	156.0
56HP to 72HP	176.0	190.0
74HP to 96HP	240.0	240.0

<Detail of Control>

Compressor:

Increase the compressor speed above the required value to return the oil to the compressor

Expansion Valve:

(In the Case of Cooling Operation) Open the expansion valve of the indoor unit under thermo-ON.

(In the Case of Heating Operation) Open the expansion valve of the outdoor unit.

<Deactivating Condition>

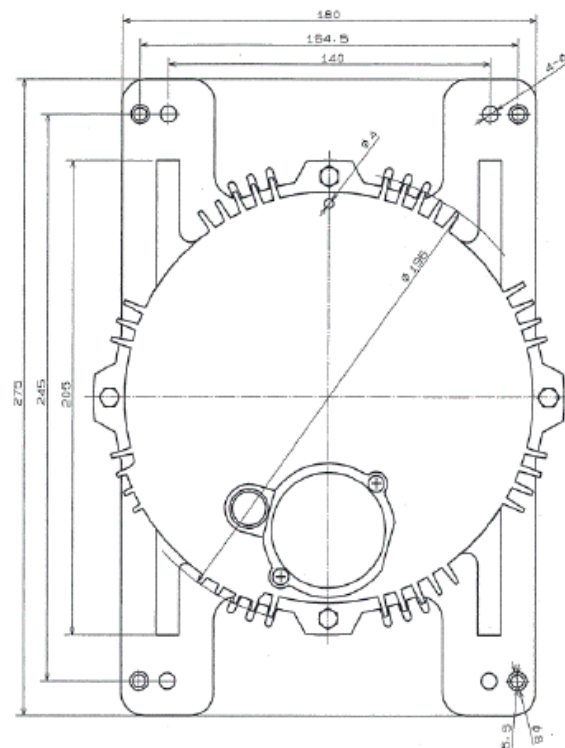
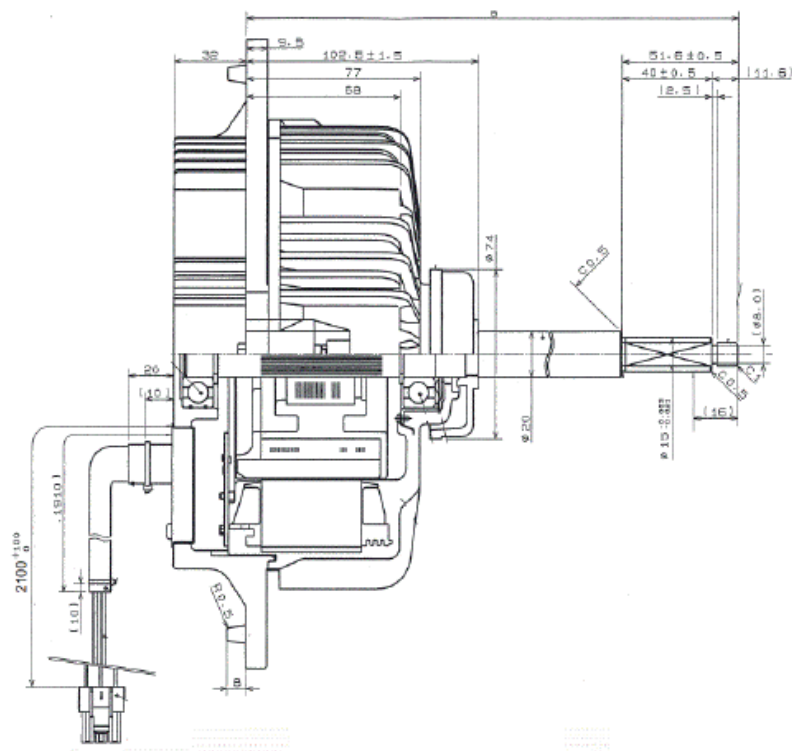
This control function is canceled when the oil return control continues for more than 60 sec. (for cooling operation) /120sec. (for heating operation).



VRF Component

Fan control and Fan RPM

■ Fan motor



Unit: mm

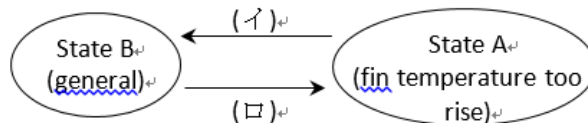
Model	a	Resistance
8HP/10HP/12HP/20HP/22HP/24HP	268.5 ± 1.5	9.2+1Ω at 20°C
14HP/16HP/18HP	227.0 ± 1.5	9.2+1Ω at 20°C

Fan motor

Fan step lower limit step: F_{omin}

Condition	HEX state	Lower limit step
State A	COND	
	EVAP	6
State B	COND	0
	EVAP	1

HP	Fomax
8HP	22
10HP	23
12HP	25
14HP	23
16HP	24
18HP	24
20HP	23
22HP	23
24HP	24



Switch condition

mark	condition
(イ)	Inverter fin temperatue : $T_{fmax_slv} \leq 67^{\circ}\text{C}$
(ロ)	Inverter fin temeratuer : $T_{fmax_slv} \geq 72^{\circ}\text{C}$ is about 5min
	or Inverter fin temeratuer $T_{fmax_slv} \geq 75^{\circ}\text{C}$

■ Fan motor

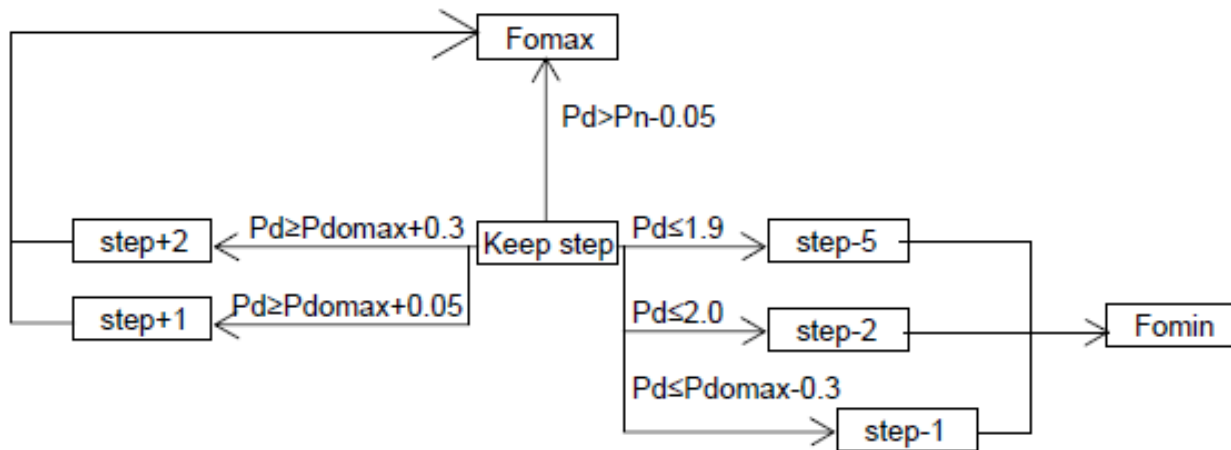
■ Start control

No	Condition	Step
1	$Ta_slv > 15[^\circ\text{C}]$	16
2	$Ta_slv \leq 15[^\circ\text{C}]$	10

Ta_slv: ambient temperature

■ Fan motor

■ Control (Cooling)



$P_n: 3.1 \text{ MPa}$

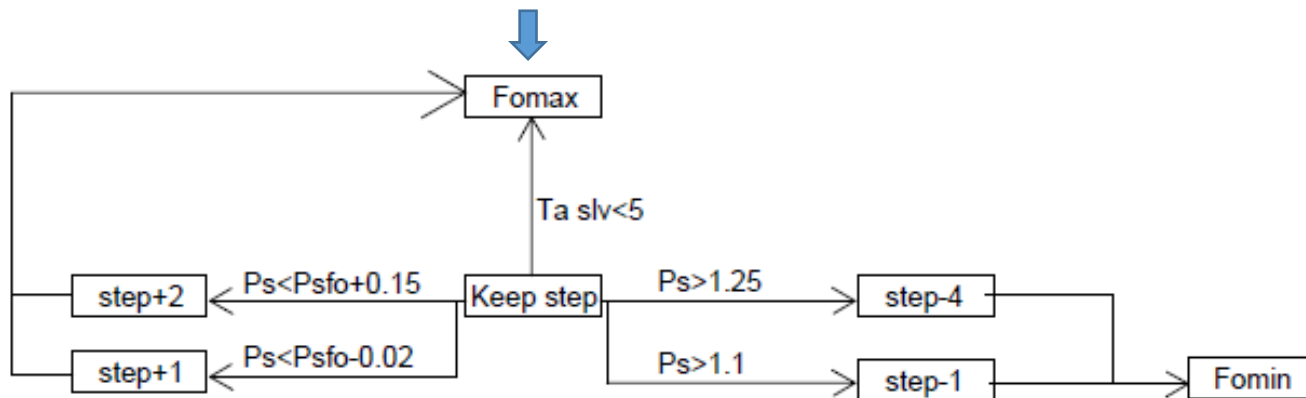
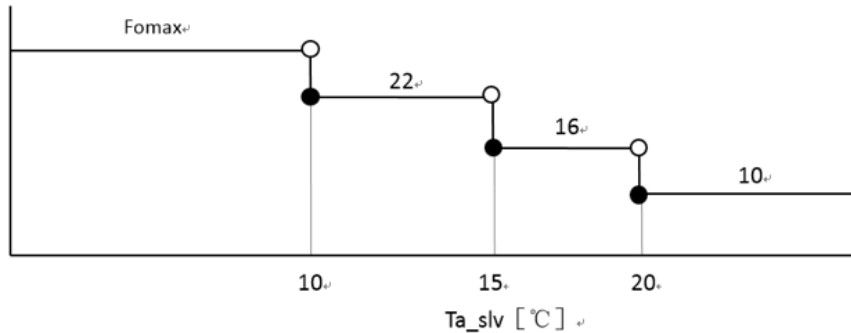
$P_{domaxc} = \{(F_{ty_slv} \times 2 + 120) \times 3 + (T_{a_slv} \times 4 + 120) \times 7\} / 1000$
($2.00 \leq P_{domaxc} \leq 3.00$)

$F_{ty_slv} = F_{slv}(n) \times 80 / \text{outdoor HP}$

$F_{slv}(n)$: instruction frequency

■ Fan motor

■ Control (Heating)

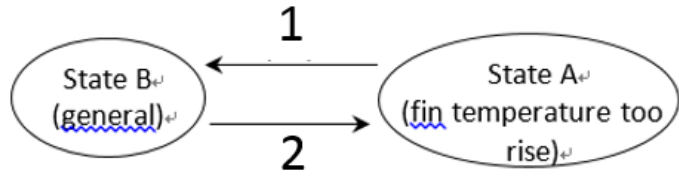


$$P_{sfo} = (760 - F_{ty_slv}) / 1000$$

$$F_{ty_slv} = F_{slv}(n) \times 80 / \text{outdoor HP}$$

$F_{slv}(n)$: instruction frequency

■ Fan motor



Switch condition

mark	condition
1	Inverter fin temperatue : $T_{fmax_slv} \leq 67^{\circ}\text{C}$
2	Inverter fin temeratuer : $T_{fmax_slv} \geq 72^{\circ}\text{C}$ is about 5min or Inverter fin temeratuer $T_{fmax_slv} \geq 75^{\circ}\text{C}$

Entrance: $T4 \leq T_{fmax_slv}$ Fan step +3

Exit: $T_{fmax_slv} < T2$

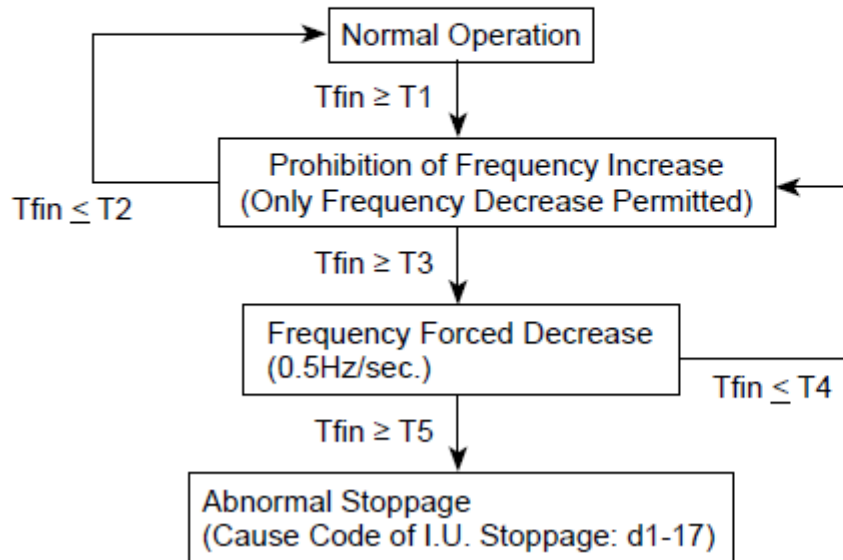
		T2	T4
State A	Generation	98	100
	Degeneration	96	98
State B	Generation	103	105
	Degeneration	101	103

■ Fan motor

P04: Inverter Fin Temperature Increase Protection Control

Inverter Fin Temperature Increase Protection Control is performed in order to prevent an inverter trip caused by a temperature increase of the inverter fin.

<Detail of Control>



Control Value

<380-415V>

[°C]

Model	T1	T2	T3	T4	T5
RAS-8.0HNBCM RAS-10HNBCM RAS-18HNBCM RAS-20HNBCM	98	96	100	98	103
RAS-12HNBCM RAS-14HNBCM RAS-16HNBCM RAS-22HNBCM RAS-24HNBCM	103	101	105	103	108

Fan motor

Fan speed and Power input

(a) Cabinet A:

		Fan speed[rpm]
		FAN1
Fan step	0	0
	1	108
	2	120
	3	132
	4	156
	5	186
	6	216
	7	228
	8	276
	9	306
	10	336
	11	372
	12	396
	13	432
	14	468
	15	516
	16	576
	17	594
	18	630
	19	666
	20	702
	21	714
	22	780
	23	804
	24	852
	25	924
	26	1002
27	1080	

(b) Cabinet B1/B2:

		Fan speed[rpm]	
		FAN1	FAN2
Fan step	0	0	0
	1	114	114
	2	126	126
	3	138	138
	4	162	162
	5	192	192
	6	222	222
	7	240	240
	8	282	282
	9	318	318
	10	348	348
	11	384	384
	12	420	420
	13	486	486
	14	576	576
	15	630	630
	16	684	684
	17	720	720
	18	786	786
	19	822	822
	20	912	912
	21	960	960
	22	1032	1032
	23	1116	1116
	24	1194	1194
	25	1230	1230
	26	1266	1266
27	1302	1302	

(c) Cabinet C1/C2:

		Fan speed[rpm]	
		FAN1	FAN2
Fan step	0	0	0
	1	108	108
	2	120	120
	3	132	132
	4	156	156
	5	186	186
	6	216	216
	7	228	228
	8	276	276
	9	306	306
	10	336	336
	11	372	372
	12	408	408
	13	456	456
	14	510	510
	15	570	570
	16	588	588
	17	618	618
	18	672	672
	19	714	714
	20	756	756
	21	798	798
	22	840	840
	23	888	888
	24	936	936
	25	984	984
	26	1032	1032
27	1080	1080	

- SVA(Gas bypass valve)

- Pd increase protection
- Ps decrease protection

Cooling:

Pd>3.6MPA,Open

Ps<0.2MPA,Open

Heating

Pd>3.5MPA,Open

Ps<0.1MPA,Open

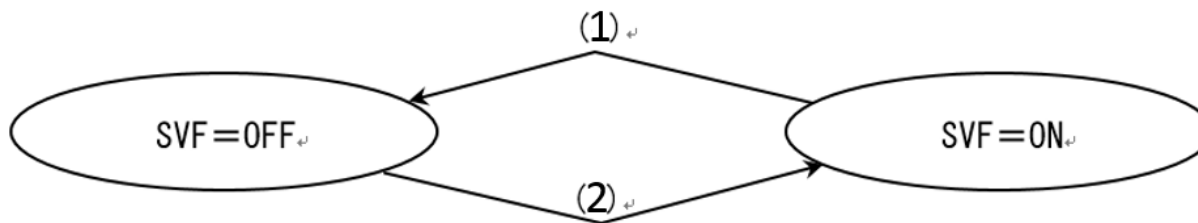
Defrosting

Closed

■ SVB

- When Comp1 Frequency≠0时, SVB=ON; otherwise, SVB=OFF.
- When Comp2 Frequency≠0时, SVC=ON; otherwise, SVB=OFF.

■ SVF



Mark	Conditions
(1) (OFF)	Meet all conditions and keep1min, SVF do the OFF operation. <ul style="list-style-type: none"> • $F_{tinv}(j) \geq 80 \times K_{comp_slv}(j)$ Hz; • COND, $20^{\circ}\text{C} \leq Ta_slv \leq 45^{\circ}\text{C}$; or EVAP, $Ta_slv \leq 15^{\circ}\text{C}$; • COND, $Pd_slv \leq Pc - 0.1$ MPa; or EVAP, $Pd_slv \leq 2.8$ MPa; • $Tdmin_slv \geq Tc_slv + 20^{\circ}\text{C}$, and $Tdave_slv(n) \geq Tdave_slv(n-1)$; or $Tdmin_slv \geq Tc_slv + 25^{\circ}\text{C}$
(2) (ON)	Meet anyone of conditions, SVF do the ON operation. <ul style="list-style-type: none"> • There have Compressor OFF; or compressor $F_{tinv}(j) \leq 40 \times K_{comp_slv}(j)$ Hz; • COND, $Ta_slv \leq 15^{\circ}\text{C}$, or $48^{\circ}\text{C} \leq Ta_slv$; EVAP, $Ta_slv \geq 18^{\circ}\text{C}$; • COND, $Pd_slv \geq 3.7$ MPa or EVAP, $Pd_slv \geq 3.2$ MPa; • $Tdmin_slv \leq Tc_slv + 15^{\circ}\text{C}$;

AA50: $K_{comp_slv} = 0.77$

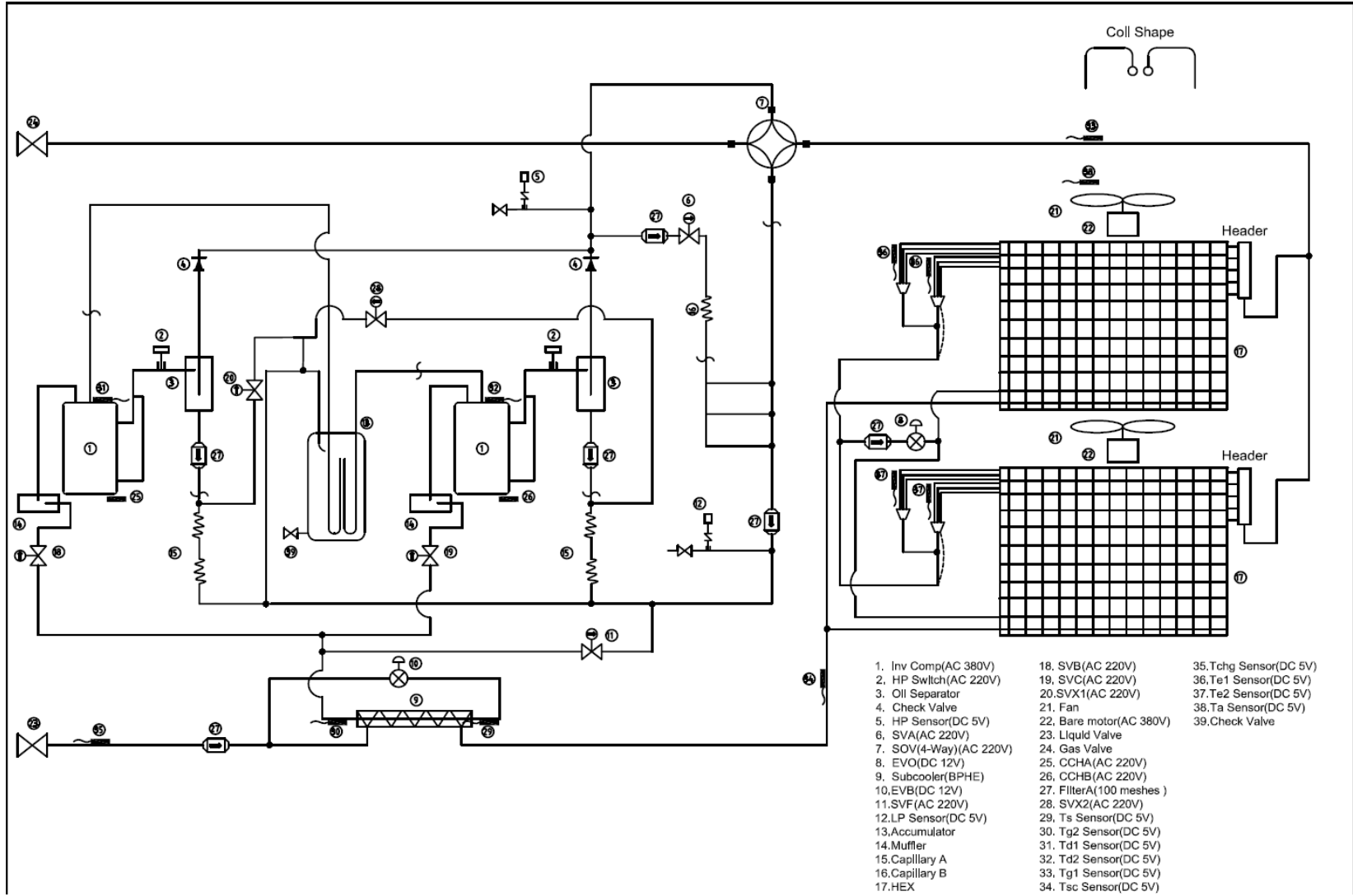
DC80: $K_{comp_slv} = 1.23$

Ta_slv: ambient temperature

Pd_slv: High pressure

Tdmin_slv: min(compressor Td), Pc=3.2

■ SVF



- | | | |
|------------------------|--------------------------|-----------------------|
| 1. Inv Comp(AC 380V) | 18. SVB(AC 220V) | 35.Tchg Sensor(DC 5V) |
| 2. HP Swltch(AC 220V) | 19. SVC(AC 220V) | 36.Te1 Sensor(DC 5V) |
| 3. Oil Separator | 20.SVX1(AC 220V) | 37.Te2 Sensor(DC 5V) |
| 4. Check Valve | 21. Fan | 38.Ta Sensor(DC 5V) |
| 5. HP Sensor(DC 5V) | 22. Bare motor(AC 380V) | 39.Check Valve |
| 6. SVA(AC 220V) | 23. Liquid Valve | |
| 7. SOV(4-Way)(AC 220V) | 24. Gas Valve | |
| 8. EVO(DC 12V) | 25. CCHA(AC 220V) | |
| 9. Subcooler(BPHE) | 26. CCHB(AC 220V) | |
| 10.EVB(DC 12V) | 27. FilterA(100 meshes) | |
| 11.SVF(AC 220V) | 28. SVX2(AC 220V) | |
| 12.LP Sensor(DC 5V) | 29. Ts Sensor(DC 5V) | |
| 13.Accumulator | 30. Tg2 Sensor(DC 5V) | |
| 14.Muffler | 31. Td1 Sensor(DC 5V) | |
| 15.Capillary A | 32. Td2 Sensor(DC 5V) | |
| 16.Capillary B | 33. Tg1 Sensor(DC 5V) | |
| 17.HEX | 34. Tsc Sensor(DC 5V) | |

■ SVX1

- When Comp1 Frequency $\neq 0 \rightarrow =0$ 时, SVX1=ON keep30s; otherwise, SV X1=OFF.

■ SVX2

- When Comp2 Frequency $\neq 0 \rightarrow =0$ 时, SVX2=ON keep30s; otherwise, SVX2=OFF

■ Electronic Expansion Valve for OU

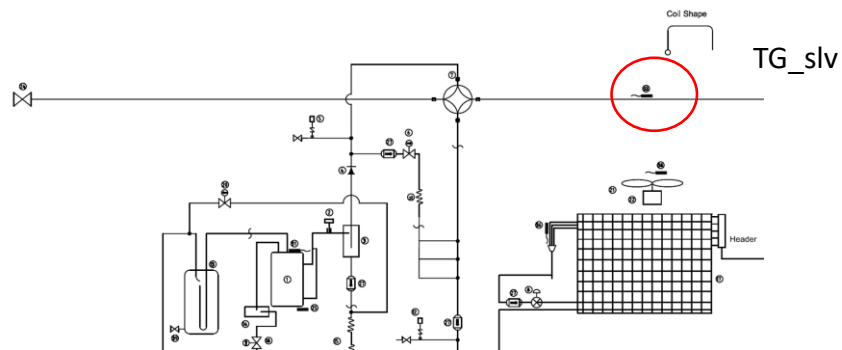
■ Cooling and defrost:

Fully open

■ Heating

OU heat exchanger SH

$$EVO(n) = EVO(n-1) \times \Delta EVOTs$$



$\Delta EVOTs$	$\Delta Ts(n)$	$-4 \leq \Delta Ts(n) < -2$	$-2 \leq \Delta Ts(n) < -1$	$-1 \leq \Delta Ts(n) < 1$	$1 \leq \Delta Ts(n) < 3$	$3 \leq \Delta Ts(n) < 6$	$6 \leq \Delta Ts(n) < 10$	$10 \leq \Delta Ts(n) < 15$	$15 \leq \Delta Ts(n)$
$8 < Q_{\Delta Ts}$	1.02	1.04	1.07	1.10	1.15	1.20	1.25	1.25	1.25
$5 < Q_{\Delta Ts} \leq 8$	1.00	1.02	1.04	1.07	1.10	1.15	1.20	1.25	1.25
$2 < Q_{\Delta Ts} \leq 5$	0.98	1.00	1.02	1.04	1.07	1.10	1.15	1.20	1.25
$0 < Q_{\Delta Ts} \leq 2$	0.95	0.98	1.00	1.02	1.04	1.07	1.10	1.15	1.20
$-1 < Q_{\Delta Ts} \leq 0$	0.90	0.95	0.98	1.00	1.02	1.04	1.07	1.10	1.15
$-3 < Q_{\Delta Ts} \leq -1$	0.85	0.90	0.95	0.98	1.00	1.02	1.04	1.07	1.10
$-6 < Q_{\Delta Ts} \leq -3$	0.80	0.85	0.90	0.95	0.98	1.00	1.02	1.04	1.07
$-9 < Q_{\Delta Ts} \leq -6$	0.80	0.80	0.85	0.90	0.95	0.98	1.00	1.02	1.04
$Q_{\Delta Ts} \leq -9$	0.80	0.80	0.80	0.85	0.90	0.95	0.98	1.00	1.02

$$Q_{\Delta Ts} : \Delta Ts(n) - \Delta Ts(n-1)$$

$$\Delta Ts(n) : TG_{slv} - Tso_{slv} - Kpl$$

$$\Delta Ts(n-1) : \text{Last } \Delta Ts(n)$$

Tso_{slv} : target evaporator temperature

$$Tso_{slv} = Tcs_{slv} + Ktsco \quad (Tso_{slv} \leq Ta_{slv})$$

Tcs_{slv} : Ps saturation temperature

$$Ktsco = ps_{slv} * 1.67 + 2.5 \quad (3-5)$$

Kpl : Pressure loss correction factor

$$Kpl = Ft_{slv} \times 0.012 - Kpl_{hosei} \quad (0 \leq Kpl \leq 3)$$

Kpl_{hosei} : Density ratio correction

$$Kpl_{hosei} = 0.8 - 0.11 \times Ta_{slv}$$

$$(0 \leq Kpl_{hosei} \leq 3)$$

■ Electronic Expansion Valve for OU(subcooling)

■ $EVB(n) = EVB(n-1) \times \Delta EVB$ ($EVB_{min} \leq EVB(n) \leq EVB_{max}$)

When SVF=OFF, $\Delta EVB = \Delta EVBTshinj$;

Other, $\Delta EVB = \Delta EVBTSc$

$Q_ \Delta Tshinj : \Delta Tshinj(n) - \Delta Tshinj(n-1)$

$\Delta Tshinj : Tshinj - Tshinj_0$

$\Delta Tshinj(n-1) : \text{last } \Delta Tshinj$

Tshinj_0: temperature difference before and after subcooling heat exchanger bypass

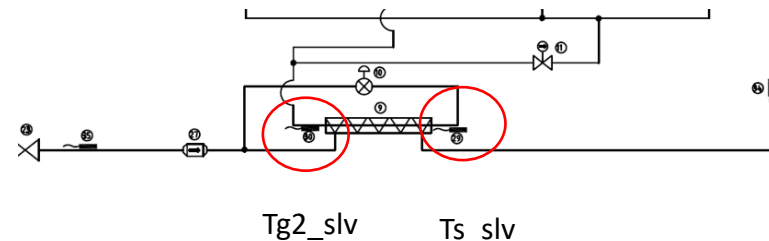
$Tshinj = Tg2_slv - Ts_slv$

Tshinj_0: target TSCch

Cooling: Tshinj_0_c: 5°C

Heating: $8 + (7 - Ta_slv) / 3$ ($8 \leq Tshinj_h \leq 13^\circ C$)

$\Delta EVBTshinj$	$\Delta Tshinj(n) \leq -9$	$-9 \leq \Delta Tshinj(n) \leq -6$	$-6 \leq \Delta Tshinj(n) \leq -4$	$-4 \leq \Delta Tshinj(n) \leq -1$	$-1 \leq \Delta Tshinj(n) \leq 3$	$3 \leq \Delta Tshinj(n) \leq 5$	$5 \leq \Delta Tshinj(n) \leq 9$	$9 \leq \Delta Tshinj(n) \leq 15$	$15 \leq \Delta Tshinj(n)$
$12 < Q_ \Delta Tshinj \leq 12$	1.01	1.02	1.04	1.06	1.08	1.1	1.12	1.15	1.2
$7 < Q_ \Delta Tshinj \leq 12$	1	1.01	1.02	1.04	1.06	1.08	1.1	1.12	1.15
$4 < Q_ \Delta Tshinj \leq 7$	0.99	1	1.01	1.02	1.04	1.06	1.08	1.1	1.12
$1 < Q_ \Delta Tshinj \leq 4$	0.96	0.99	1	1.01	1.02	1.04	1.06	1.08	1.1
$-3 < Q_ \Delta Tshinj \leq 1$	0.93	0.96	0.99	1	1.01	1.02	1.04	1.06	1.08
$-5 < Q_ \Delta Tshinj \leq -3$	0.9	0.93	0.96	0.99	1	1.01	1.02	1.04	1.06
$-7 < Q_ \Delta Tshinj \leq -5$	0.85	0.9	0.93	0.96	0.99	1	1.01	1.02	1.04
$-9 < Q_ \Delta Tshinj \leq -7$	0.85	0.85	0.9	0.93	0.96	0.99	1	1.01	1.02
$Q_ \Delta Tshinj \leq -9$	0.8	0.8	0.85	0.9	0.93	0.96	0.99	1	1.01



■ Electronic Expansion Valve for OU(subcooling)

■ $EVB(n) = EVB(n-1) \times \Delta EVB$ ($EVB_{min} \leq EVB(n) \leq EVB_{max}$)

When SVF=OFF, $\Delta EVB = \Delta EVBT_{shinj}$;

Other, $\Delta EVB = \Delta EVBT_{Sc}$

$Q_{\Delta TSc} : \Delta TSc(n) - \Delta TSc(n-1)$

$\Delta TSc(n) : TSc_{cho} - TSc_{ch}$

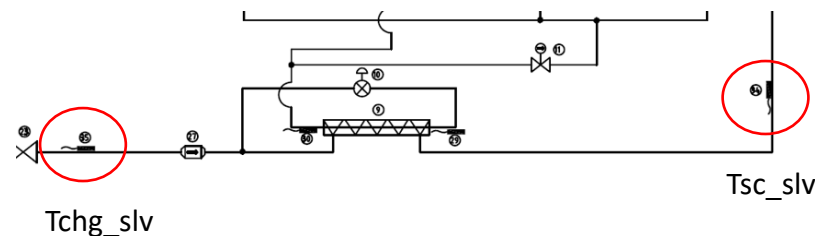
$\Delta TSc(n-1) : \text{last } \Delta TSc(n)$

TScch: temperature difference before and after subcooling heat exchanger

Cooling; $TSc_{ch} = TSc_{slv} - Tch_{g_slv}$

Heating; $TSc_{ch} = Tch_{g_slv} - TSc_{slv}$

TSccho: target TScch



$\Delta EVBT_{Sc}$	$\Delta TSc(n) < -9$	$\Delta TSc(n) < -6$	$\Delta TSc(n) < -3$	$\Delta TSc(n) < -1$	$\Delta TSc(n) < 2$	$\Delta TSc(n) < 4$	$\Delta TSc(n) < 7$	$\Delta TSc(n) < 10$	$\Delta TSc(n)$
$7 < Q_{\Delta TSc}$	1.00	1.01	1.03	1.06	1.10	1.13	1.17	1.20	1.20
$5 < Q_{\Delta TSc} \leq 7$	0.98	1.00	1.01	1.03	1.06	1.10	1.13	1.17	1.20
$3 < Q_{\Delta TSc} \leq 5$	0.96	0.98	1.00	1.01	1.03	1.06	1.10	1.13	1.17
$1 < Q_{\Delta TSc} \leq 3$	0.93	0.96	0.98	1.00	1.01	1.03	1.06	1.10	1.13
$-2 < Q_{\Delta TSc} \leq 1$	0.89	0.93	0.96	0.98	1.00	1.01	1.03	1.06	1.10
$-4 < Q_{\Delta TSc} \leq -2$	0.84	0.89	0.93	0.96	0.99	1.00	1.01	1.03	1.06
$-6 < Q_{\Delta TSc} \leq -4$	0.80	0.84	0.89	0.93	0.96	0.98	1.00	1.00	1.01
$-8 < Q_{\Delta TSc} \leq -6$	0.80	0.80	0.84	0.89	0.93	0.96	0.98	1.00	1.00
$Q_{\Delta TSc} \leq -8$	0.80	0.80	0.80	0.84	0.89	0.93	0.93	0.98	1.00

■ Electronic Expansion Valve for OU(subcooling)

■ Cooling

priority	Condition	TSCcho
①	$TSC_{slv} - TSCcho_c < Tcs_{slv} + 9$	$TSC_{slv} - (Tcs_{slv} + 9)$
②	$TSC_{slv} - TSCcho_c < Ticmax - 7$	$TSC_{slv} - (Ticmax - 7)$
③	Other	$TSCcho_c$

Ticmax : the max inlet temperature of indoor unit which is turned on

TSCcho_c: $TSCcho_c = 8 + (Fty_{slv} + 20) \times 3/30 + (Ta_{slv} - 33) \times 5/15$

■ Heating

priority	Condition	TSCcho
①	$Tchg_{slv} - TSCcho_h < 2$	$Tchg_{slv} - 2$
②	$Tchg_{slv} - TSCcho_h < Ta_{slv} + 3$	$Tchg_{slv} - (Ta_{slv} + 3)$
③	Other	$TSCcho_h$

$Fty_{slv} = Fslv(n) \times 80 / \text{outdoor HP}$

Fslv(n): instruction frequency

Ta_slv: ambient temp

TSCcho_h: $TSCcho_h = \{(Fty_{slv} + 20) \times 4/30 + (Ta_{slv} - 10) \times 2/15 + \beta\} \times Kacc$

$Pd_{slv} \leq 2.20$ and $Ps_{slv} \leq 0.30$ and $\beta \leq -5$ Kacc=0

Other, Kacc=1

$\beta = 0.5 \times SCave(n) - 7.5$ ($-7 \leq \beta \leq 0$)

SCave: the average of SC(indoor units which is turned on)

■ Electronic Expansion Valve for IU

■ Cooling :

$$EVI(n) = EVI(n-1) \times \Delta EVIC \quad (90 \leq EVIC(n) \leq EVI_{max})$$

$\Delta EVIC$	$\Delta SH(n) < -3$	$-3 \leq \Delta SH(n) < -2$	$-2 \leq \Delta SH(n) < -1$	$-1 \leq \Delta SH(n) < 1$	$1 \leq \Delta SH(n) < 3$	$3 \leq \Delta SH(n) < 6$	$6 \leq \Delta SH(n) < 10$	$10 \leq \Delta SH(n) < 15$	$15 \leq \Delta SH(n)$
$7 < Q_{\Delta SH}$	1.01	1.03	1.06	1.10	1.13	1.17	1.20	1.20	1.20
$4 < Q_{\Delta SH} \leq 7$	1.00	1.01	1.03	1.06	1.10	1.13	1.17	1.20	1.20
$2 < Q_{\Delta SH} \leq 4$	0.98	1.00	1.01	1.03	1.06	1.10	1.13	1.17	1.20
$1 < Q_{\Delta SH} \leq 2$	0.96	0.98	1.00	1.01	1.03	1.06	1.10	1.13	1.17
$-2 < Q_{\Delta SH} \leq 1$	0.93	0.96	0.98	1.00	1.01	1.03	1.06	1.10	1.13
$-3 < Q_{\Delta SH} \leq -2$	0.89	0.93	0.96	0.99	1.00	1.01	1.03	1.06	1.10
$-5 < Q_{\Delta SH} \leq -3$	0.84	0.89	0.93	0.96	0.98	1.00	1.01	1.03	1.06
$-8 < Q_{\Delta SH} \leq -5$	0.80	0.84	0.89	0.93	0.96	0.98	1.00	1.01	1.03
$Q_{\Delta SH} \leq -8$	0.80	0.80	0.84	0.89	0.93	0.96	0.98	1.00	1.01

$Q_{\Delta SH} : \Delta SH(n) - \Delta SH(n-1)$

$\Delta SH(n) : SH - SH_0 \quad (-20 \leq \Delta SH(n))$

$\Delta SH(n-1) : \text{last } \Delta SH(n)$

$SH = Trg(k) - TrL(k) + \text{correction value}$

$Trg(k) : \text{IDU gas temp } [^{\circ}C]$

$TrL(k) : \text{IDU liquid temp } [^{\circ}C]$

correction value : IDU SH correction value

■ Electronic Expansion Valve for IU

■ Cooling :

$$\begin{aligned} \text{Sho:target SH} \\ = \text{SHon} + \text{SHo_hosei1} + \text{SHo_hosei2} + \text{SHo_hosei3} \\ (0 \leq \text{SHo} \leq 8) \end{aligned}$$

$$\text{SHon} = 0$$

$$\text{SHo_hosei1} = \text{KSHo1} - \text{KSHo1min}$$

$$\text{KSHo1}; \text{KSHo1} = \{[(\text{Ts} + \text{Ts_rev}) - \text{Ti}] \times 4 + 80\} / 10$$

$$\text{KSHo1min}: \min(\text{KSHo1})$$

$$\text{SHo_hosei2} = (\text{KSHo2} - \text{KSHo2min}) \times 10 / 30$$

$$\text{KSHo2}; \text{KSHo2} = \text{KSHo2} = \{\text{Ti} \times 4 + (\text{Ts} + \text{Ts_rev}) \times 6 - 120\} / 10$$

$$\text{KSHo2min}: \min(\text{KSHo2})$$

$$\text{SHo_hosei3}$$

Condition	SHo hosei3
$60 < \text{SHHz}$	0
$30 < \text{SHHz} \leq 60$	1
Other	2

$$\text{SHHz} = \text{Ft} \times 100 / (\text{Outdoor capacity HP} \times 10 / 80)$$

$$/ \{(\text{indoor unit power on number} + 5) / 3\} / 5$$

Ts: setting tem

Ts_rev: Ts correction

Ti: inlet tem

■ Electronic Expansion Valve for IU

■ Heating :

$$\Delta EVI = [Kp \times \{\Delta SC(n) - \Delta SC(n-1)\} + Ki \times \Delta SC(n) \times T] \times Krtevi \quad (\Delta EVI_{min} \leq \Delta EVI \leq \Delta EVI_{max})$$

$$EVI(n) = EVI(n-1) + \Delta EVI \quad (EVI_{min_hot} \leq EVI(n) \leq EVI_{max})$$

EVI(n) : now

EVI(n-1) : last

ΔEVI : EXV change pls

$\Delta SC(n)$: $SC - SC_o$ ($-10 \leq \Delta SC(n)$)

$\Delta SC(n-1)$: last $SC(n)$

$SC = T_c - TrL(k) + \text{correction value}$

T_c : Saturation temperature (Pdmax) [°C]

$TrL(k)$: IDU liquid temp [°C]

correction value : IDU SC correction value

$Kp=1, Ki=3$

T : time

$Krtevi$: indoor EXV correction

SC_o : target SC [°C]

$SC_o = SC_{oset}$

$SC_{oset} = SC_{ave} + K_{as}$

SC_{ave} : the average of SC (turn on indoor units)

K_{as} : Indoor SC correction

$K_{as} = \{(\Delta T_{1h_{ave}} - \Delta T_{1h(k)}) \times 4\} / (\Delta T_{1h_{max}} - \Delta T_{1h_{min}})$ ($-1 \leq K_{as} \leq 1$)

$\Delta T_{1h(k)}$: inlet temperature and T_s difference

$\Delta T_{1h_{ave}}$: ΔT_{1h} average

$\Delta T_{1h_{max}}$: ΔT_{1h} max

$\Delta T_{1h_{min}}$: ΔT_{1h} min

Indoor capacity	$Krtevi$
<56	1
>63	3

■ Sub cooling heat exchanger

H3/8" (J-Bimetal-08)

H1/2" (J-Bimetal-09)

Technical requirement:
 1: Connector tolerance refer to separated detail connector drawings
 2: Tolerances not specified here are all +/2 mm

Design Pressure(PS)	4.5MPa(45bar)
Testing Pressure(N ₂)	4.5MPa(45bar)
Design Temperature(TS)	-196/+200 °C
Leak Testing	He
Volume(L)	0.17L/H1H2Side; 0.15L/H3H4Side
Weight(kg)	1.24Kg
Danfoss Code	
Label Code	F-MP-113

Mrk	ECR/Comments	Date	Projection	Scale	Size	Material	Bolt vertically :±2°	
DRN				2:1	A4	Plate material:	±2°	
CHK						Connection:	304L	
MFG						Connector:	304L	
APP						Cover plate:	304L	
Danfoss Standard: 500B0598								
Designation								
C22L-E-16-								
H1H2(H3/8")/								
H3H4(H1/2")/2-M6*20/								
Installation Drawing								
Confidential: Property of Danfoss A/S Nordborg, Denmark. Not to be handed over to copied or used by third party. Two or three dimensional reproduction of contents to be authorized by Danfoss A/S.								
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							2018.05.09	

■ Outdoor Unit control

Control Device	Control				
	Cooling Operation *		Heating Operation		Defrosting
	Control Category	Purpose of Control	Control Category	Purpose of Control	Condition
Inverter Frequency of Compressor	Total I.U. Operating Capacity	Inverter Frequency Control is carried out to make I.U. air inlet temperature to temperature setpoint.	Total I.U. Operating Capacity	Inverter Frequency Control is carried out to make I.U. air inlet temperature to temperature setpoint.	All of the compressors: ON
Electronic Expansion Valve for O.U. Heat Exchanger	Capacity Control	Fully open	O.U. Heat Exchanger SH	PI control is carried out to achieve the targeted value of O.U. heat exchanger SH.	Fully open
Electronic Expansion Valve for Supercooling Heat Exchanger (Injection OFF)	Tsc - Tchg	PI Control is carried out to achieve the target value of Tsc - Tchg.	Tchg - Tsc	PI Control is carried out to achieve the target value of Tchg - Tsc.	Tsc - Tchg
Electronic Expansion Valve for Supercooling Heat Exchanger (Injection ON)	Tg ₂ - Ts	PI Control is carried out to achieve the target value of Tg ₂ - Ts.	Tg ₂ - Ts	PI Control is carried out to achieve the target value of Tg ₂ - Ts.	Tg ₂ - Ts
Electronic Expansion Valve for I.U. Heat Exchanger	I.U. Heat Exchanger SH	PI control is carried out to achieve the targeted value of I.U. heat exchanger SH.	I.U. Heat Exchanger SC	Controls supercooling of I.U. liquid thermistor to achieve the targeted value.	I.U. Heat Exchanger SH Control
Outdoor Fan	Pd Control	PI control is carried out to achieve the targeted value of Pd.	Ps Control	PI control is carried out to achieve the targeted value of Ps.	Stop
Gas Bypass Valve (SVA)	1. Pd Increase Protection 2. Ps Decrease Protection	1. Pd>3.6MPa: Open 2. Ps<0.2MPa: Open	1. Pd Increase Protection 2. Ps Decrease Protection	1. Pd>3.5MPa: Open 2. Ps<0.1MPa: Open	Closed

Pd: Discharge Pressure
 Ps: Suction Pressure
 SH: Superheat
 Tsc: Subcooler Temperature
 Tchg: Liquid Stop Valve Temperature
 I.U.: Indoor Unit
 O.U.: Outdoor Unit

(*): Dry operation is included in the cooling operation.



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Questions?



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THANK YOU